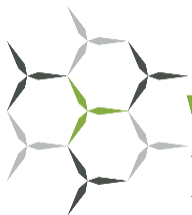
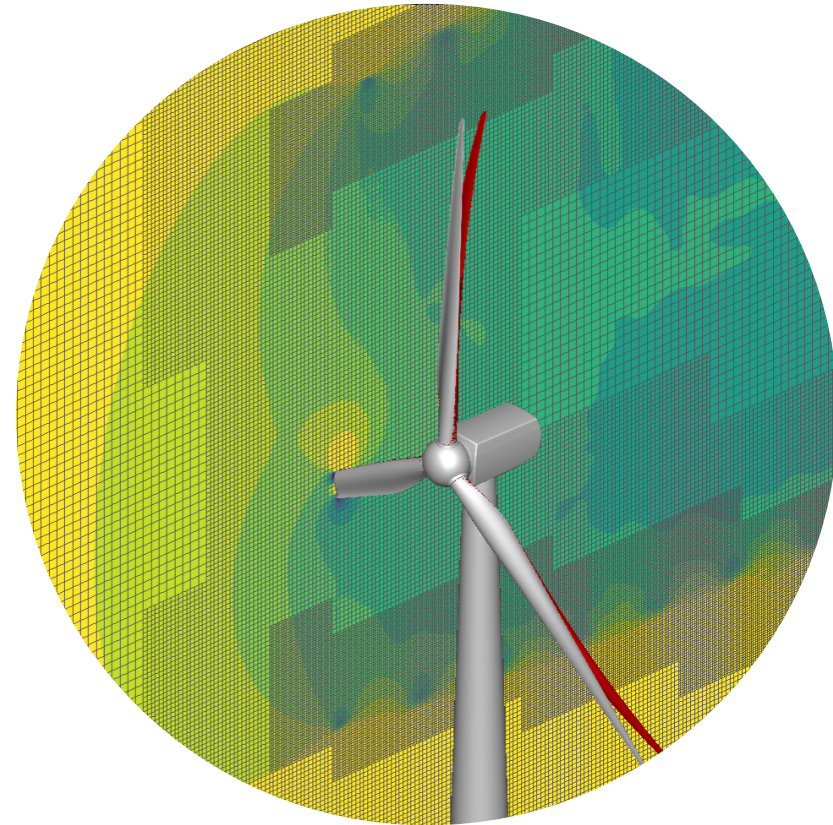


CFD for Wind Turbine Engineering Problems

Thorsten Lutz

G. Bangga, G. Guma, L. Klein, P. Letzgus,
C. Schulz, F. Seel, P. Weiing, A. Wolf, P. Zamre
and further former PhD students

lutz@iag.uni-stuttgart.de



WINDForS

Windenergie Forschungscluster
Wind Energy Research Cluster



Deutscher Akademischer Austausch Dienst
German Academic Exchange Service



Overview

Introduction to CFD

- CFD basics & mesh generation
- BEM vs. CFD
- Advanced CFD chain

Exemplary CFD Applications

- Noise prediction & reduction
- Dynamic stall & vortex generators
- Rotor & wake flow physics
- Aeroelasticity & low frequency noise
- Complex & urban terrain

Introduction to CFD

Introduction to CFD Motivation

- Why do we need CFD simulations in Wind Turbine Design?
 - Loads and power output are influenced by
 - 3D viscous effects, aeroelasticity
 - Blade-tower interaction, wake impact, unsteady aerodynamic effects
 - atmospheric boundary layer interaction, complex topography
 - Provides detailed information of the 3D flow field which can be used for an aerodynamic and aeroacoustic evaluation
 - In CFD calculation fewer simplifying assumptions are needed compared to traditional engineering models like BEM

 **More accurate in prediction of turbine behavior**

Introduction to CFD The Navier-Stokes-Equations

$$\frac{\partial}{\partial t} \iiint_V \underline{W} dV + \oint_S (\underline{F}_c + \underline{F}_v) dS = \iiint_V \underline{Q} dV$$

- This basic equation can be derived with the assumption of a finite control volume
 - In this control volume the change with respect to time of a flow variable φ is the sum of:
 - Change of φ by Convection into the control volume
 - Change of φ by Diffusion into the control volume
 - Production of φ within the control volume
- The Navier-Stokes Equation is the fundamental equation for viscous flow analysis

Introduction to CFD The Navier-Stokes-Equations

- There are different methods to solve the Navier-Stokes Equation and to simulate a (turbulent) flow around a rigid body
 - **DNS** **D**irect **N**umerical **S**imulation
 - The Navier Stokes Equation is solved directly
 - A large amount of grid points is needed
 - Flow simulations with DNS are only possible for low Reynolds Numbers

Introduction to CFD The Navier-Stokes-Equations

- **LES** **L**arge **E**ddy **S**imulation
 - Only large turbulence scales are resolved, therefore the range of length scales is limited (→ low pass filtering)
 - Small scales are described by a subgrid scale model
 - Less computational time is needed according to DNS
- **(U)RANS** **(U**nsteady) **R**eynolds-**A**veraged **N**avier-**S**tokes Equations
 - State of the art for complex turbulent flow simulations
 - A flow quantity is decomposed into a time-averaged and a fluctuating quantity
 - With the averaging, models for the description of turbulence are needed (closure problem)

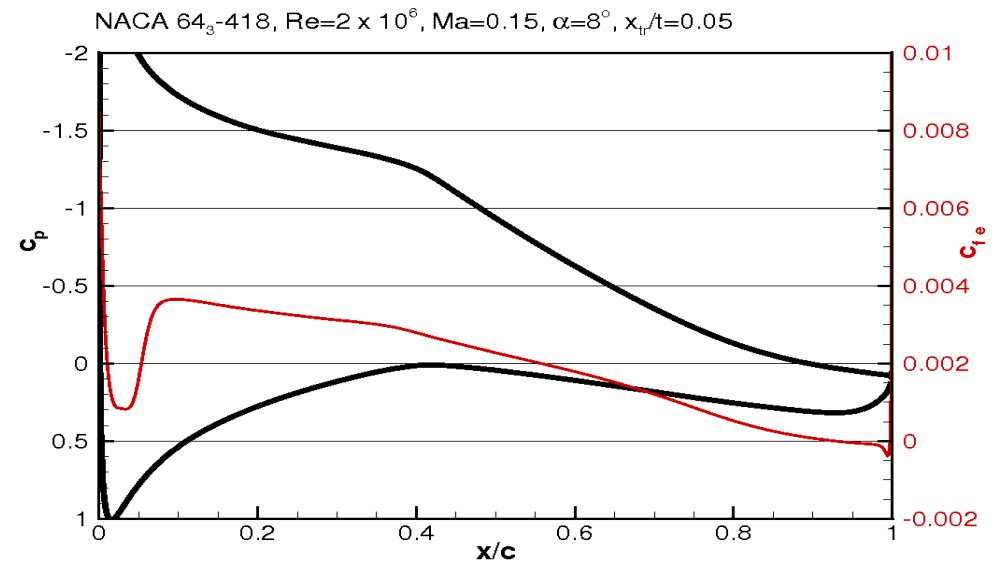
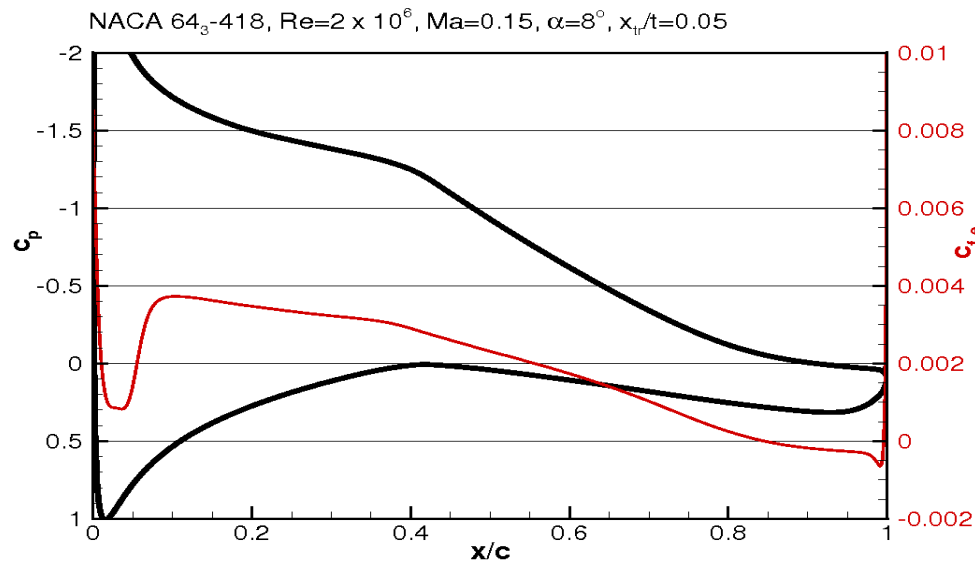
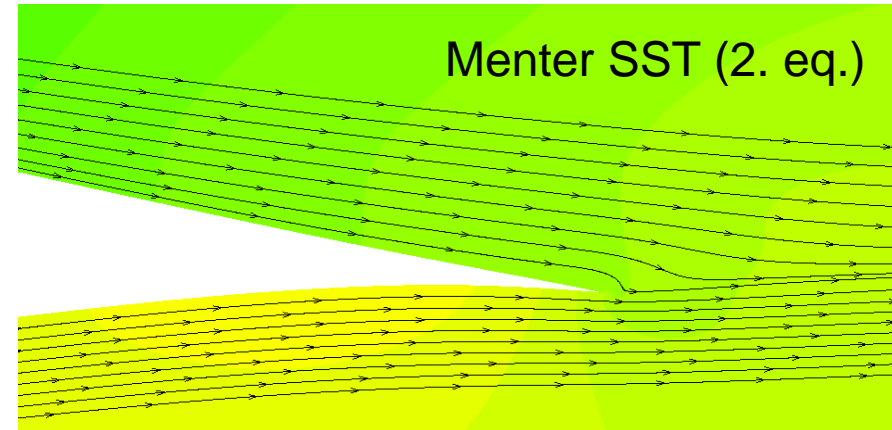
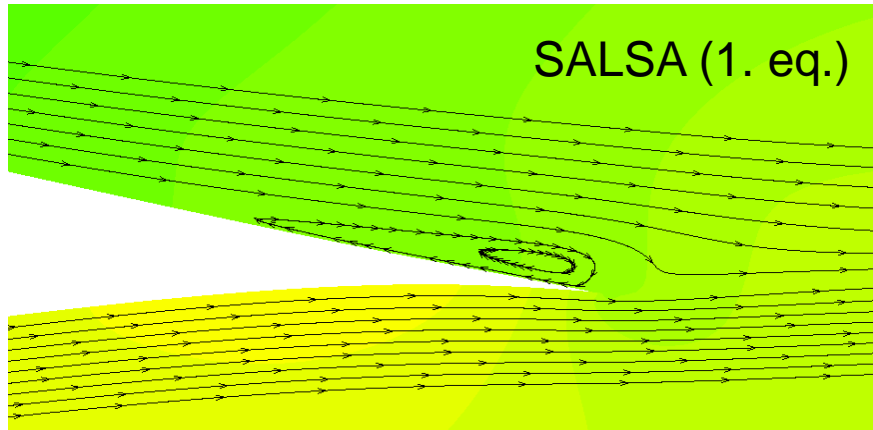
Introduction to CFD Turbulence Modelling in RANS

- Boussinesq introduced the concept of the eddy viscosity ν_t
 - Turbulent Stresses are related to the mean flow

$$\tau_{ij}^F = -\rho \overline{u_i' u_j'} = 2\rho \nu_t \overline{S_{ij}} - \frac{2}{3} \rho k \delta_{ij}$$

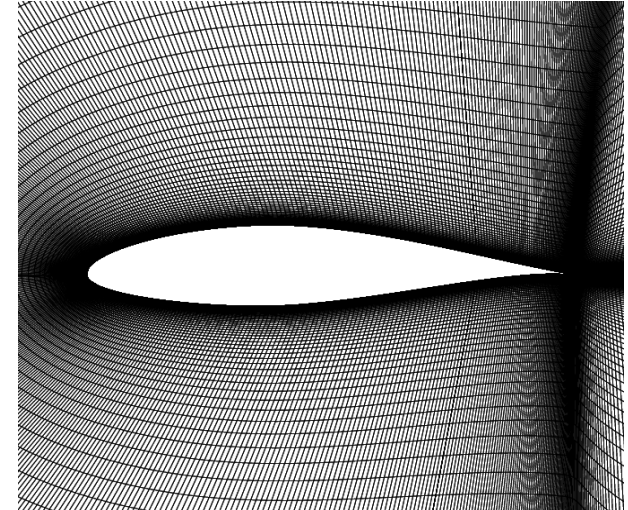
- Many turbulence models are based on this assumption
 - Zero-Equation Models: Calculation of eddy viscosity by an algebraic equation
 - One-Equation Models: Solving one transport equation for the viscosity quantity
 - Two-Equation Models: Solving two transport equations e.g. for the turbulent kinetic energy k and the turbulent dissipation ϵ

Introduction to CFD Turbulence Modelling in RANS

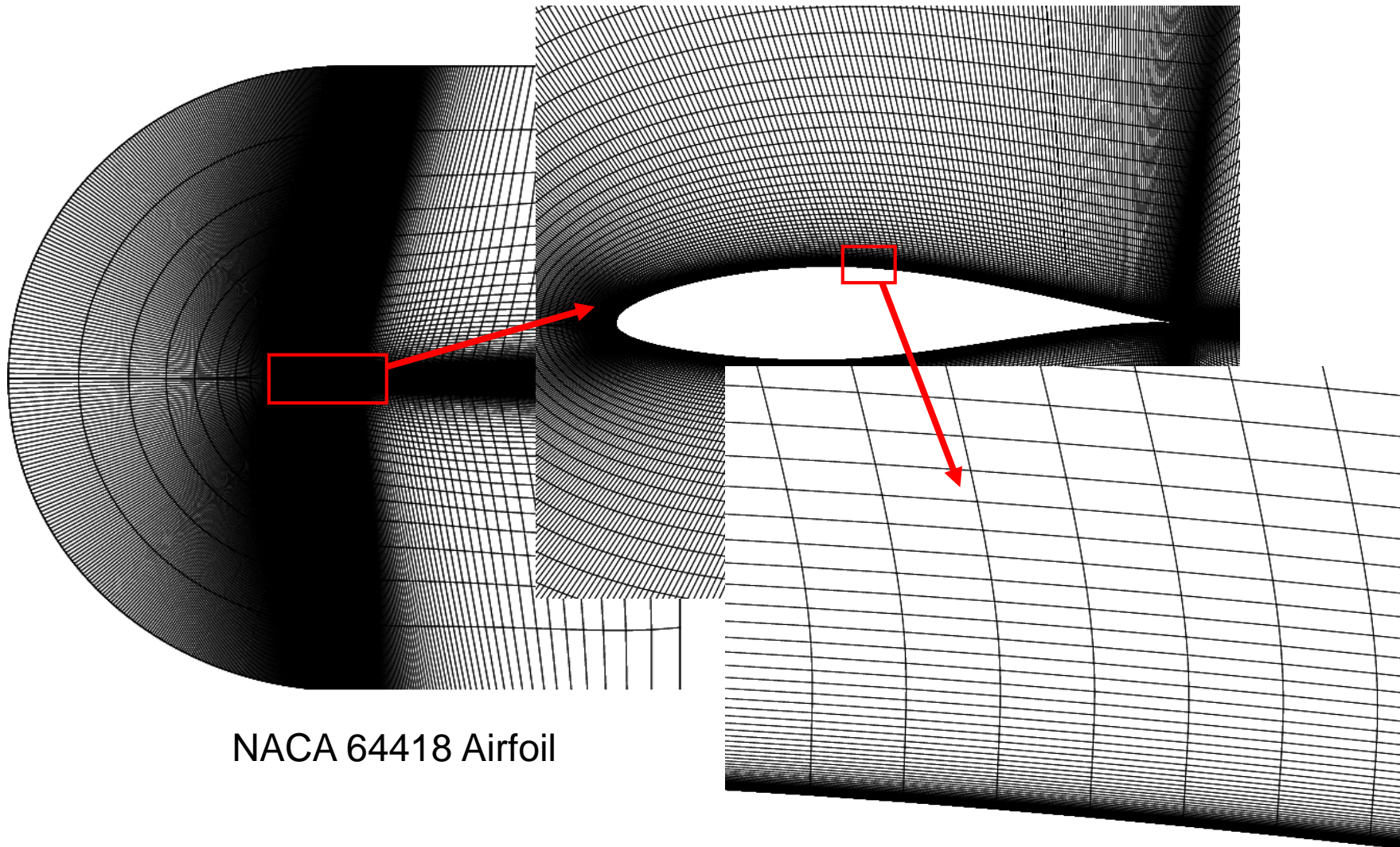


Introduction to CFD Grid Generation

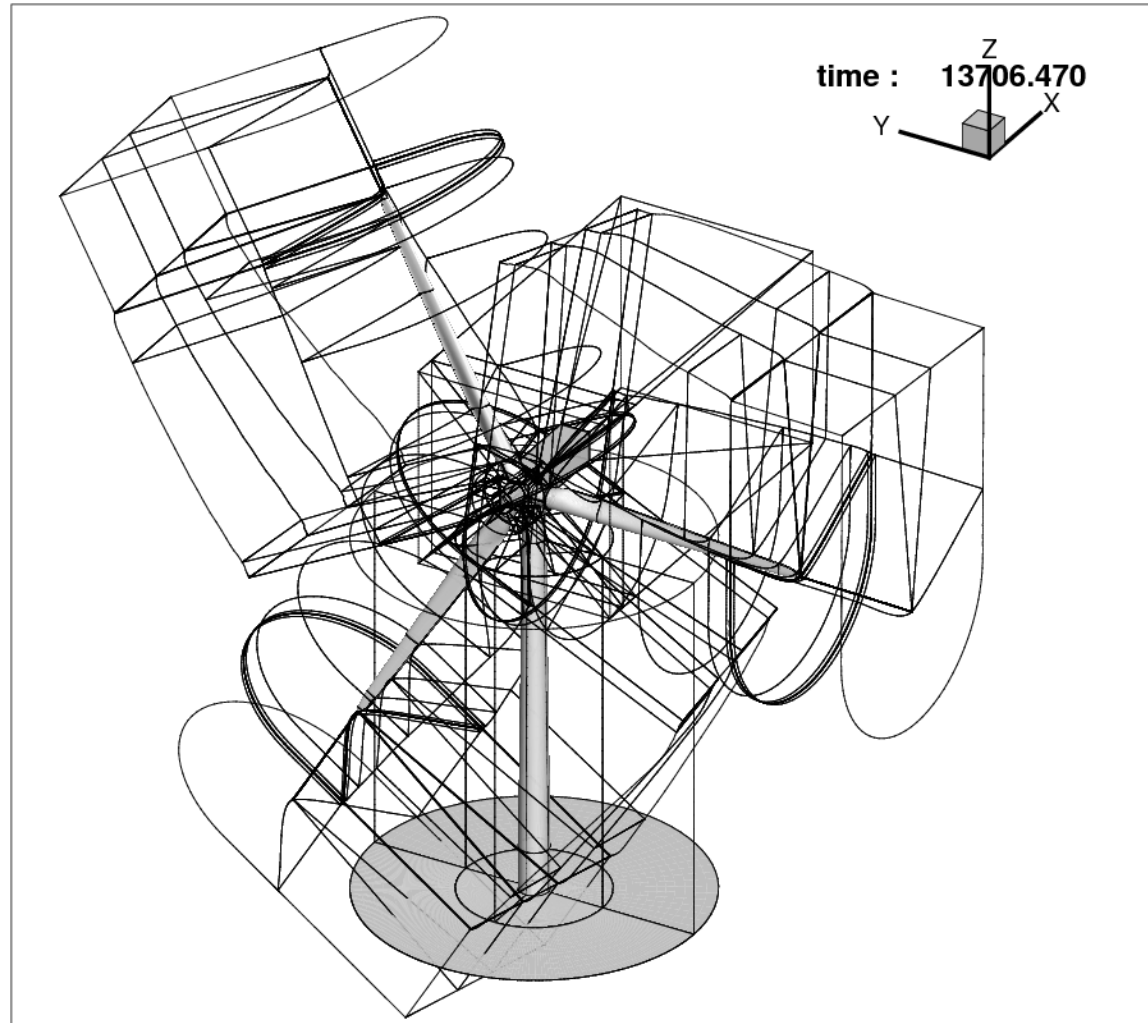
- The Navier-Stokes Equations have to be solved numerically
 - Different discretisation techniques are used
 - Finite difference
 - Finite element
 - **Finite volume (most common)**
 - Lattice Boltzmann Methods
 - Meshes are build around the geometry and fill the flow domain
 - There should be a significant distance between the outer boundaries of the flow and the geometry to reduce errors from reflection at the outer boundaries



Introduction to CFD Grid Generation - Structured Grids 2D



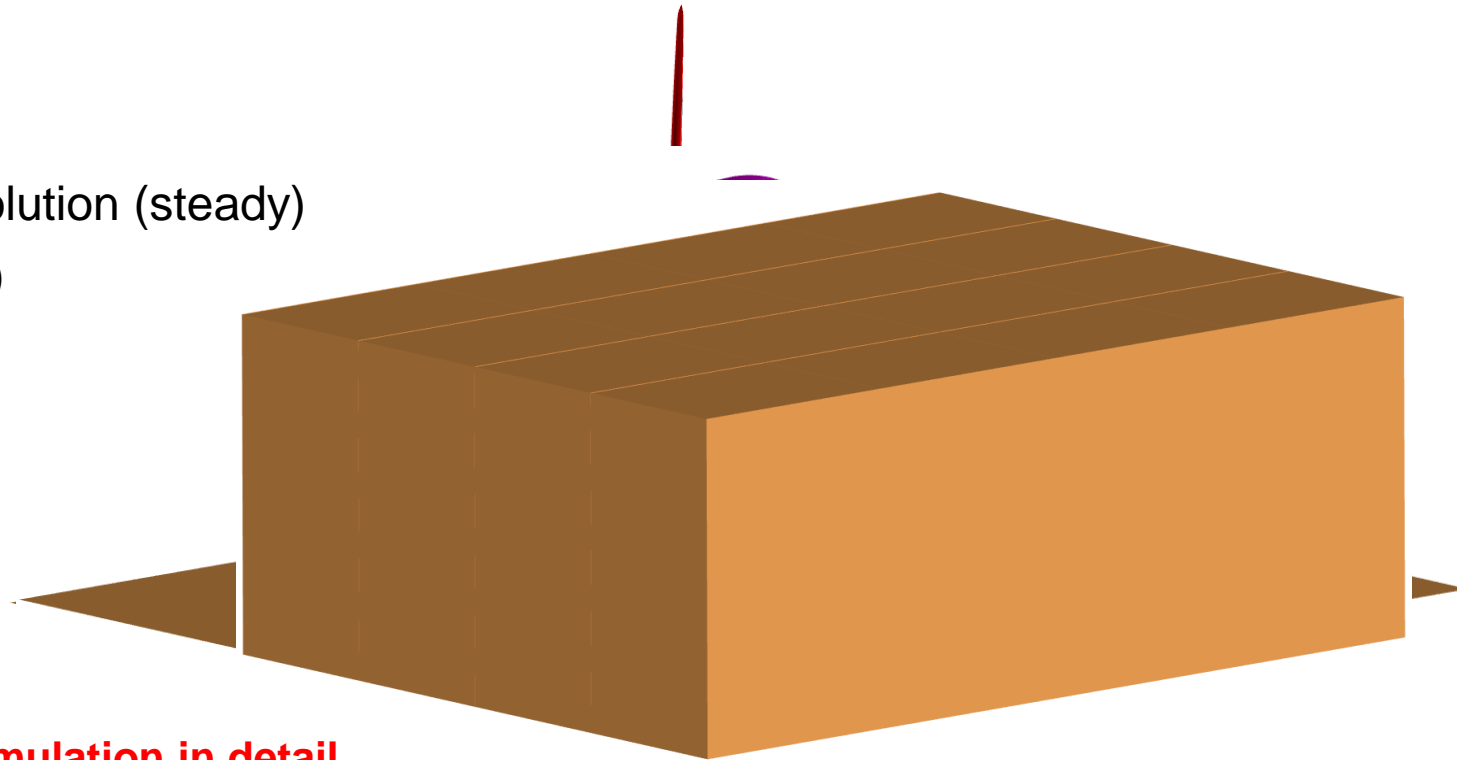
Introduction to CFD Grid Generation- Structured Grids 3D



Introduction to CFD Grid Generation- Structured Grids 3D

A wind turbine simulation consists of several meshes for:

- Spinner (rotating)
- Nacelle (steady)
- Tower (steady)
- Blades (rotating)
- Better tip vortex resolution (steady)
- Background (steady)



**Wind turbine simulation in detail
affords meshing of several components**

Introduction to CFD Computational Efforts for WT Simulation

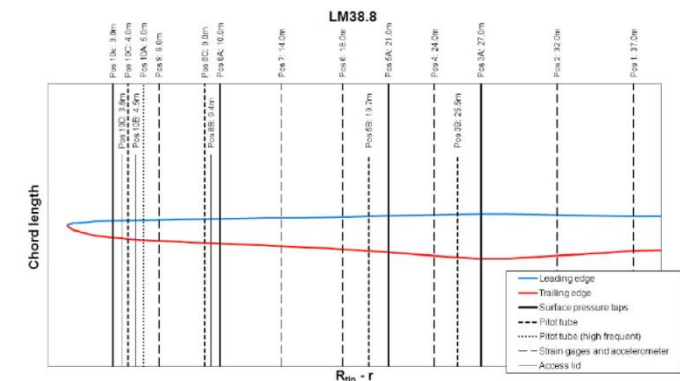
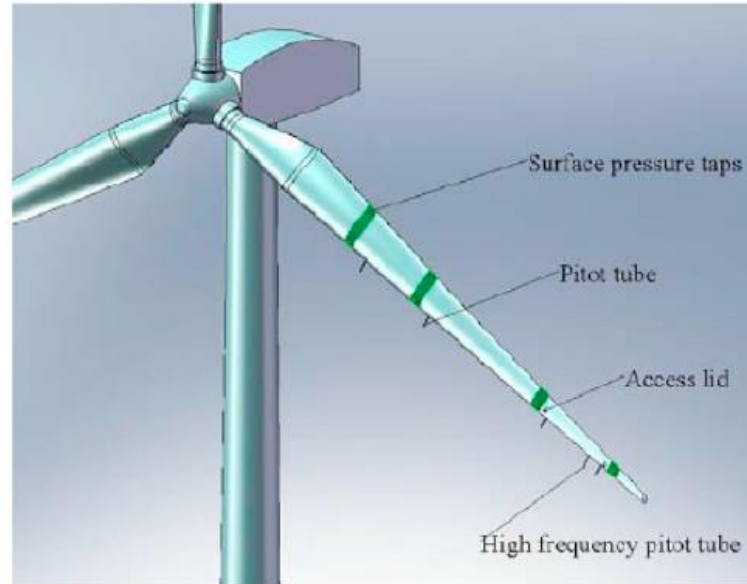
	1/3 Model (normal)	Full Model (simple)	Full Model (detailed)
Background + VortexGrid	16.7 Mio	66 Mio	97 Mio
Tower	-	1.0 Mio	1.0 Mio
Nacelle	0.8 Mio	2.5 Mio	2.5 Mio
Spinner	0.6 Mio	1.7 Mio	1.7 Mio
Blade	8.8 Mio	8.8 Mio x 3	8.8 Mio x 3
Sum	26.9 Mio	97.6 Mio	128.6 Mio
Time step	5°	5°	1°

Depending on the flow situation a minimum of 10 revolutions is recommended

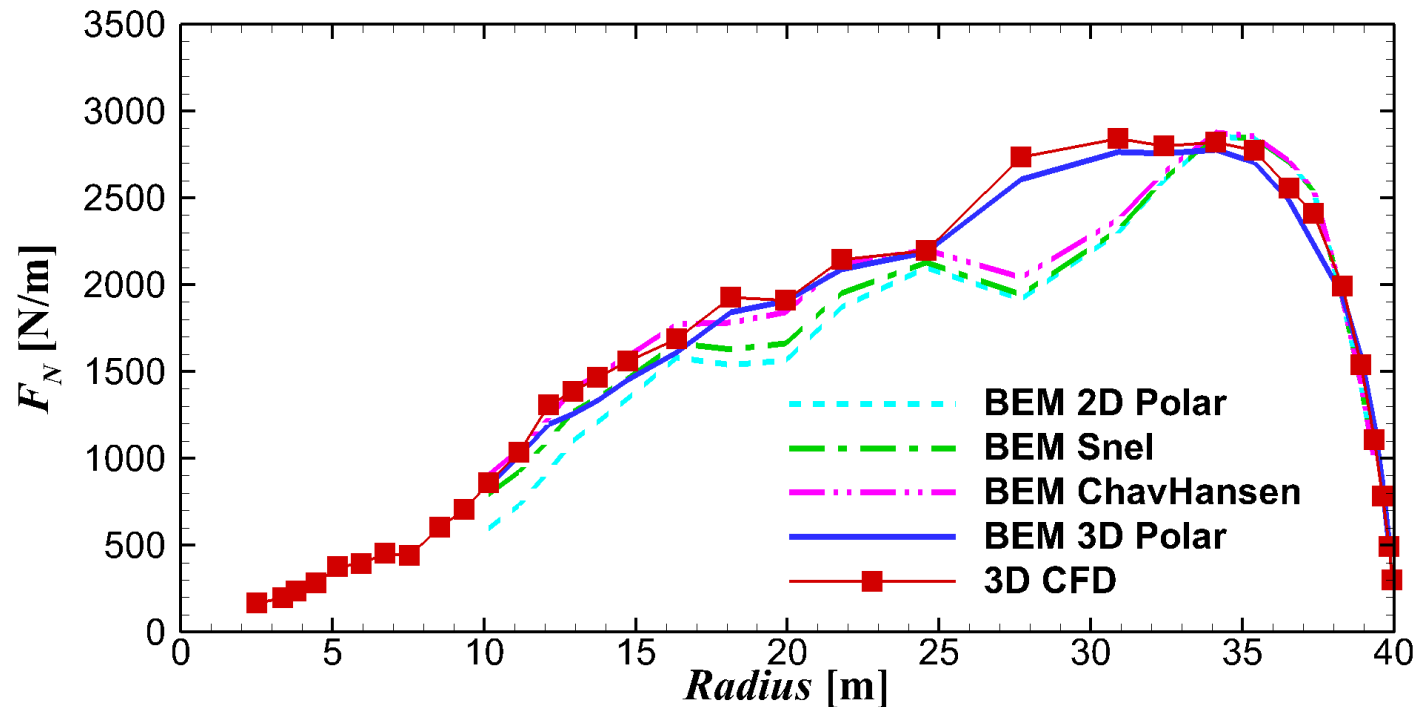
BEM vs. CFD DanAero Turbine

- Turbine has a diameter of 80 m, tilt 5° and 1.4 m prebend at the tip
- The field measurement was conducted from 2007-2010 at Tjaereborg, 1 km from the north sea
- The nominal rated power is 2.3 MW
- The turbine is investigated within IEA Task 29 Phase IV project

Madsen et al. The DAN-AERO MW experiments: final report. Danmarks tekniske universitet, risø nationallaboratoriet for bæredygtig energi. 2010. available at: orbit.dtu.dk. [Accessed 13 November 2017].



BEM vs. CFD Results – DanAero Turbine

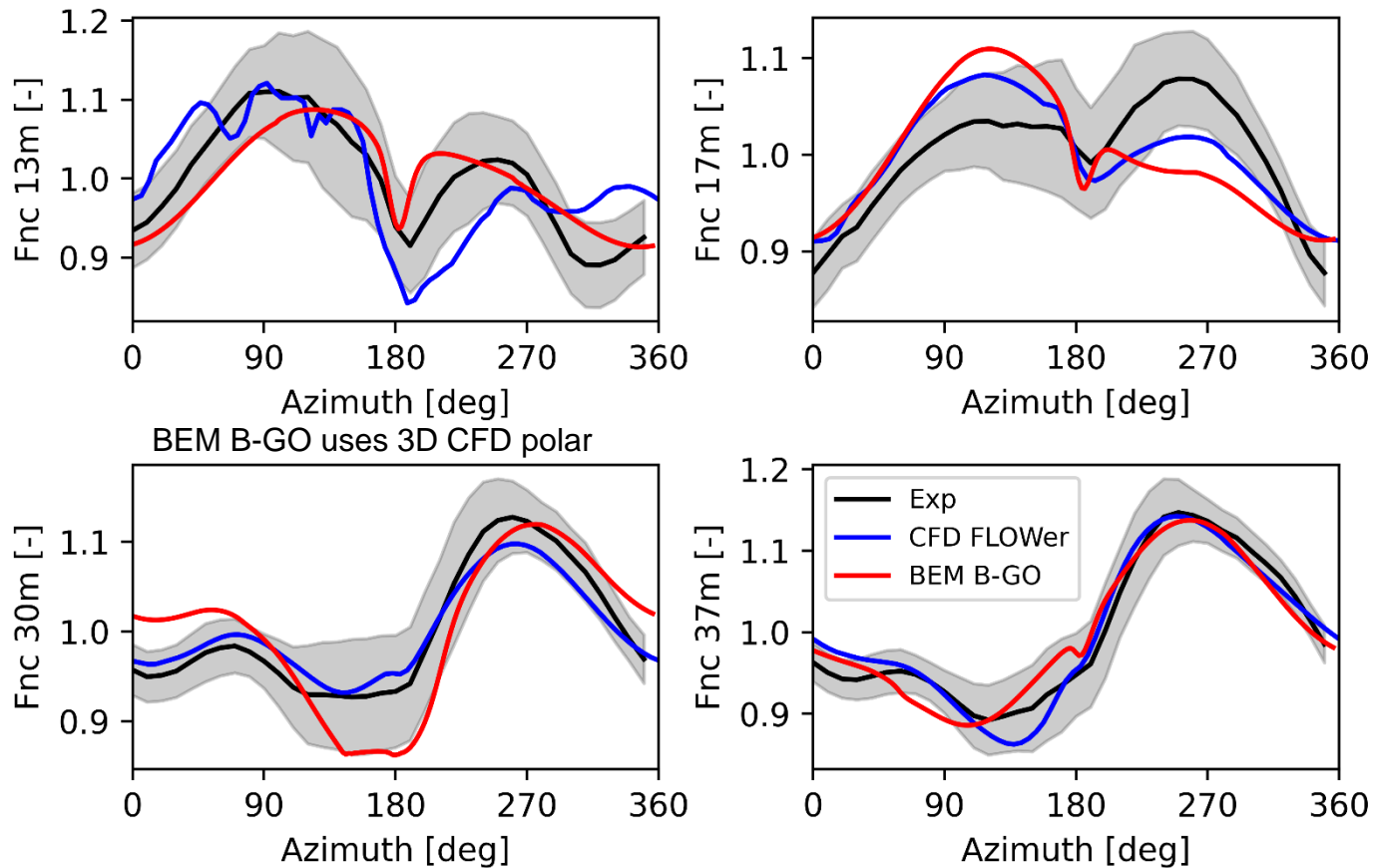


CFD extracted 3D polar data improves BEM prediction for a 2.3 MW DanAero turbine!

G. Bangga

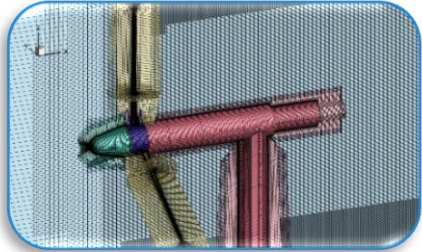
BEM vs. CFD Results – DanAero Turbine

Also in complex flow condition under 38° yaw misalignment!

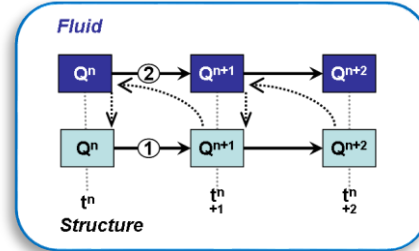


G. Bangga

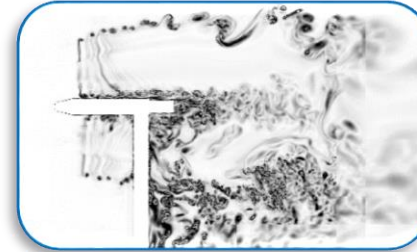
Advanced CFD Chain USTUTT-IGAG Chain Based on CFD Solver FLOWer



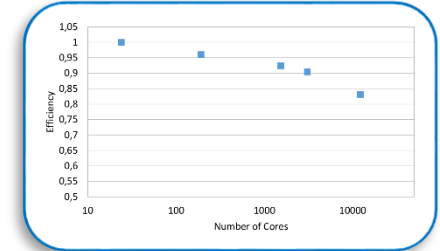
Moving, deformable grids



Fluid-structure interaction

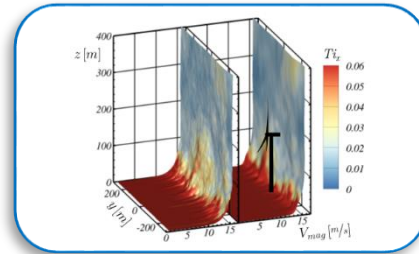


Higher-order numerics

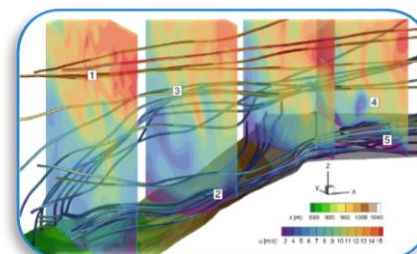


Efficient parallelization

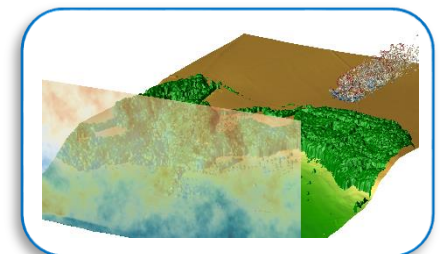
Aerodynamics, loads and noise emission of the controlled, flexible turbine in atmospheric inflow



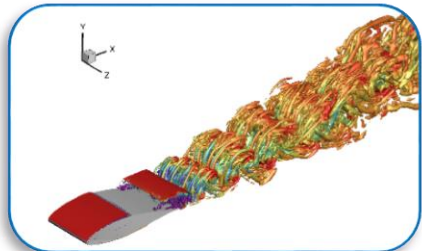
Inflow turbulence & shear



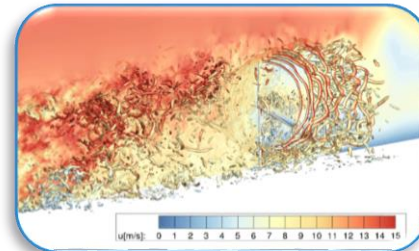
Complex terrain, vegetation



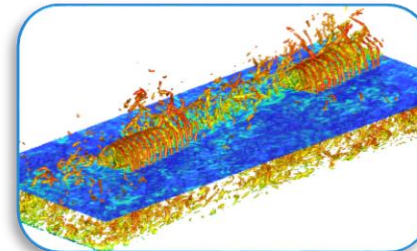
Stratification



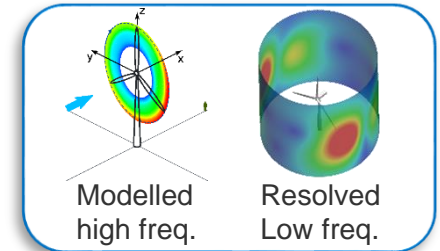
Airfoil aerodynamics



Rotor aerodynamics

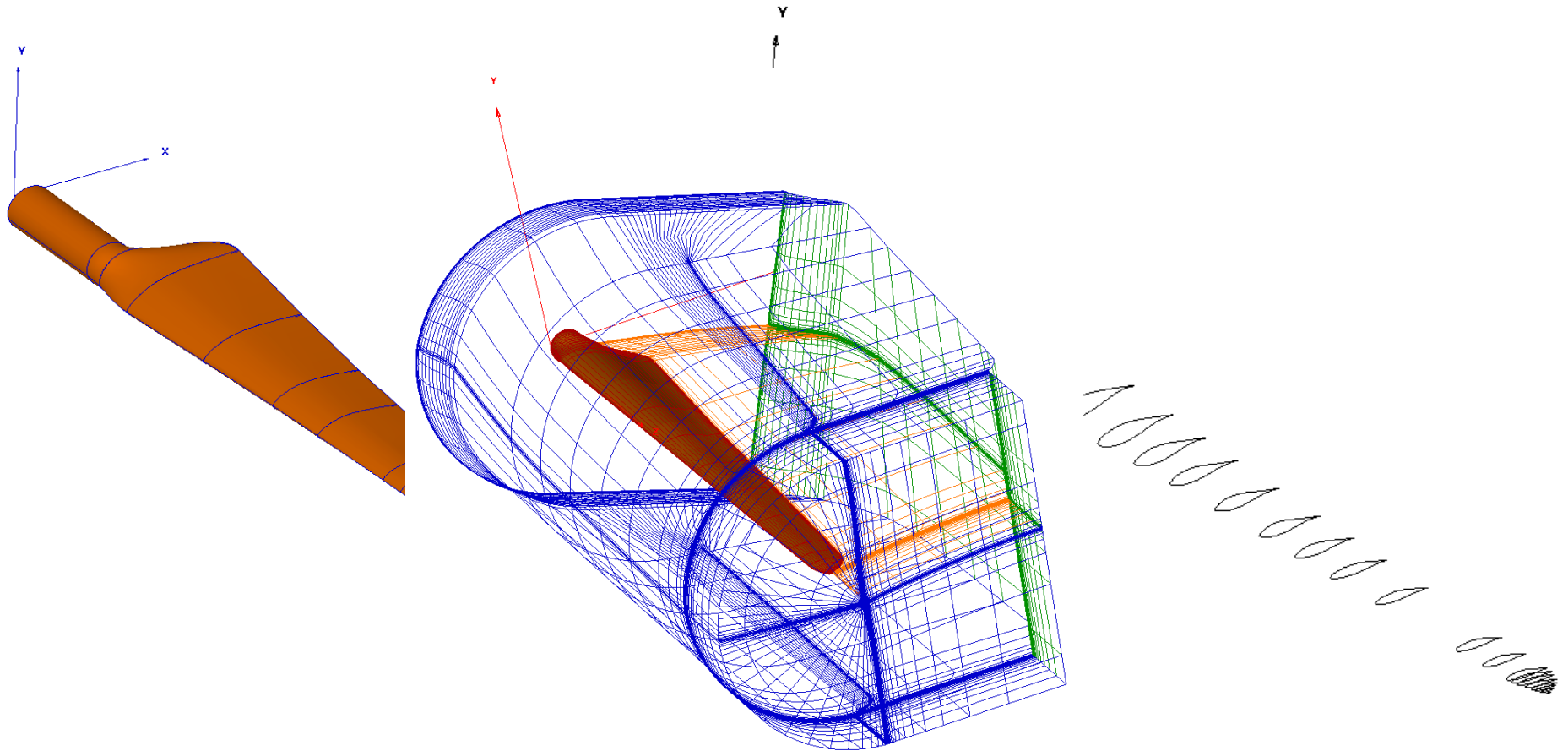


Wake interactions



Aeroacoustics

Advanced CFD Chain Automated Blade Meshing



CAD data in iges format

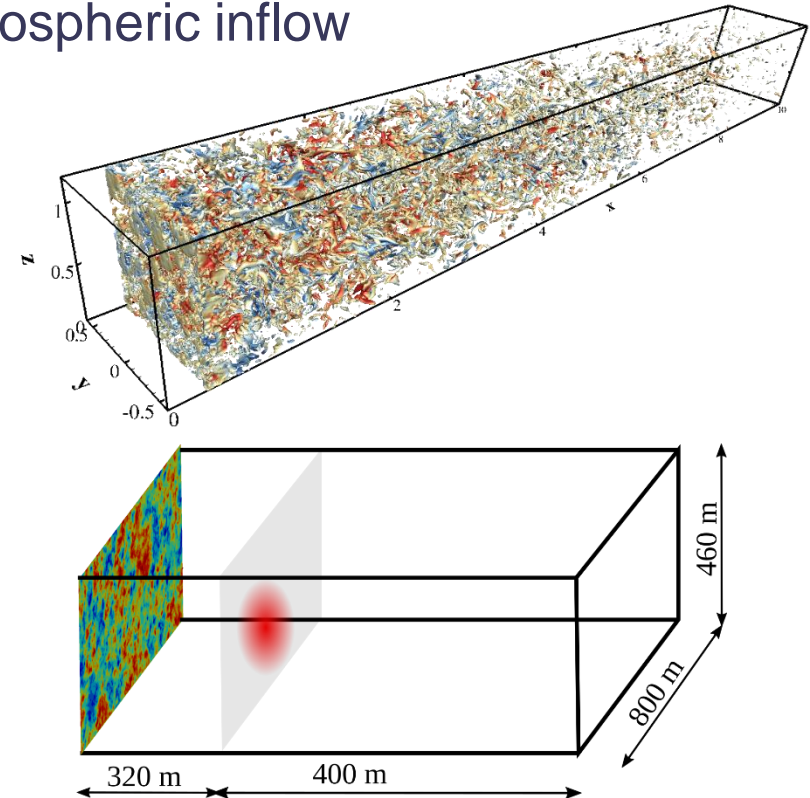
Meshed blade

Cuts of the blade
(actual basis of mesh generation)

Advanced CFD Chain Unsteady atmospheric inflow

As a forcing term

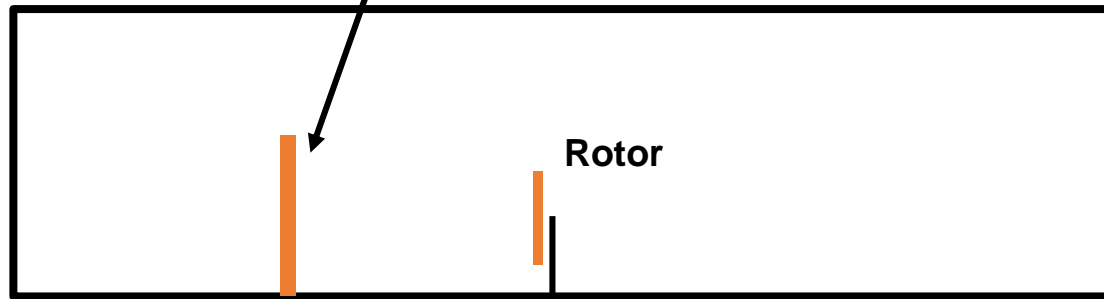
- Turbulent flow is fed into the simulation domain a forcing term in three directions, using only its fluctuations
- Mean velocity profile is specified at the inflow domain
- Both are combined by means of superposition
- Minimizing numerical dissipation



Mean vel profile

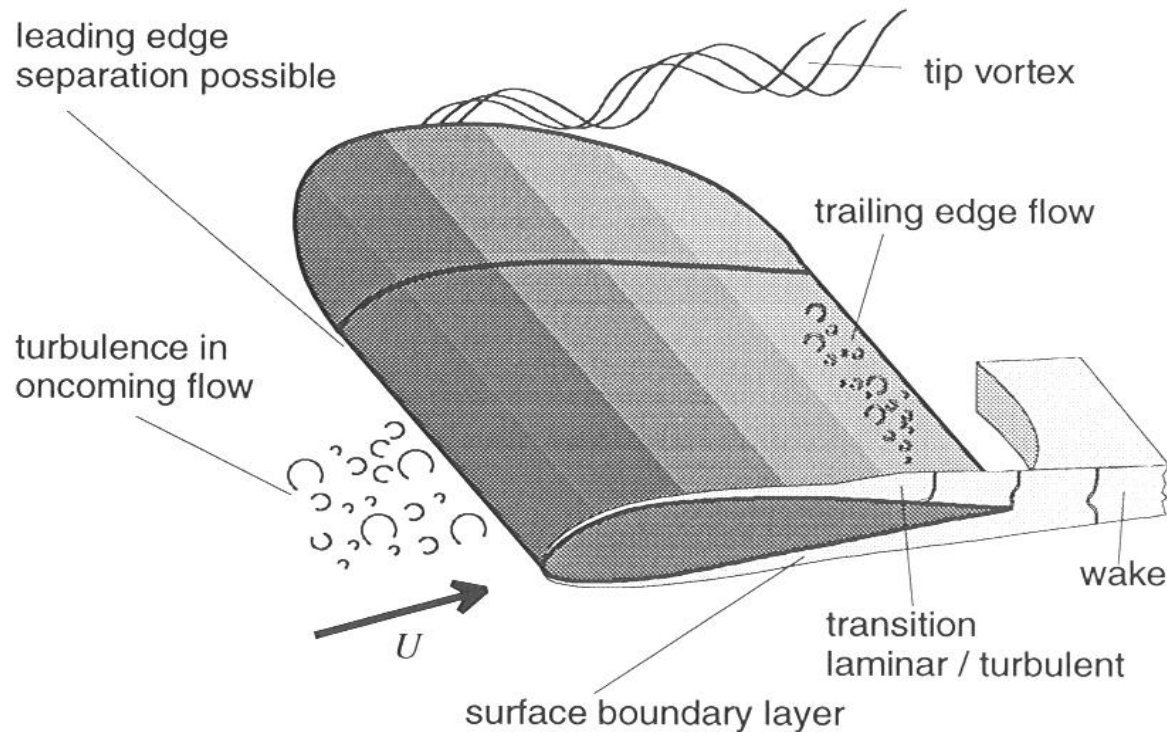
Turb fluctuations

Rotor

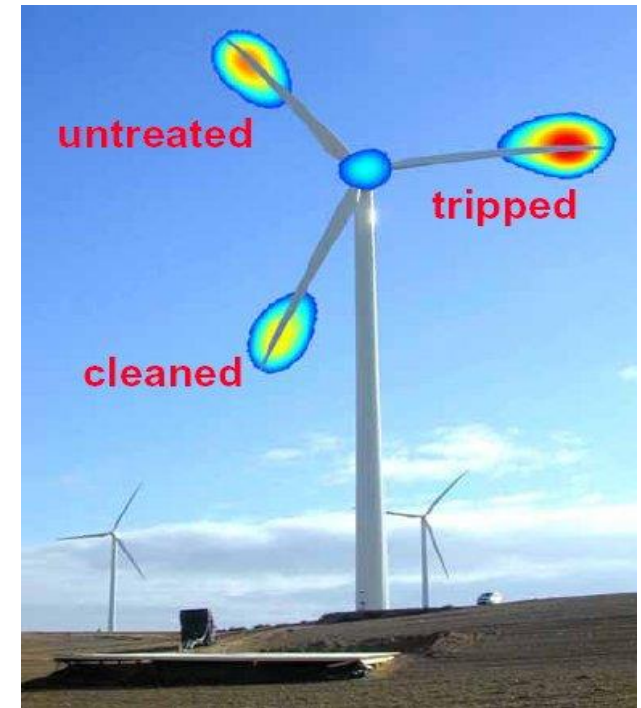


Exemplary CFD Applications

Noise Prediction & Reduction Relevant Aeroacoustic Noise Sources

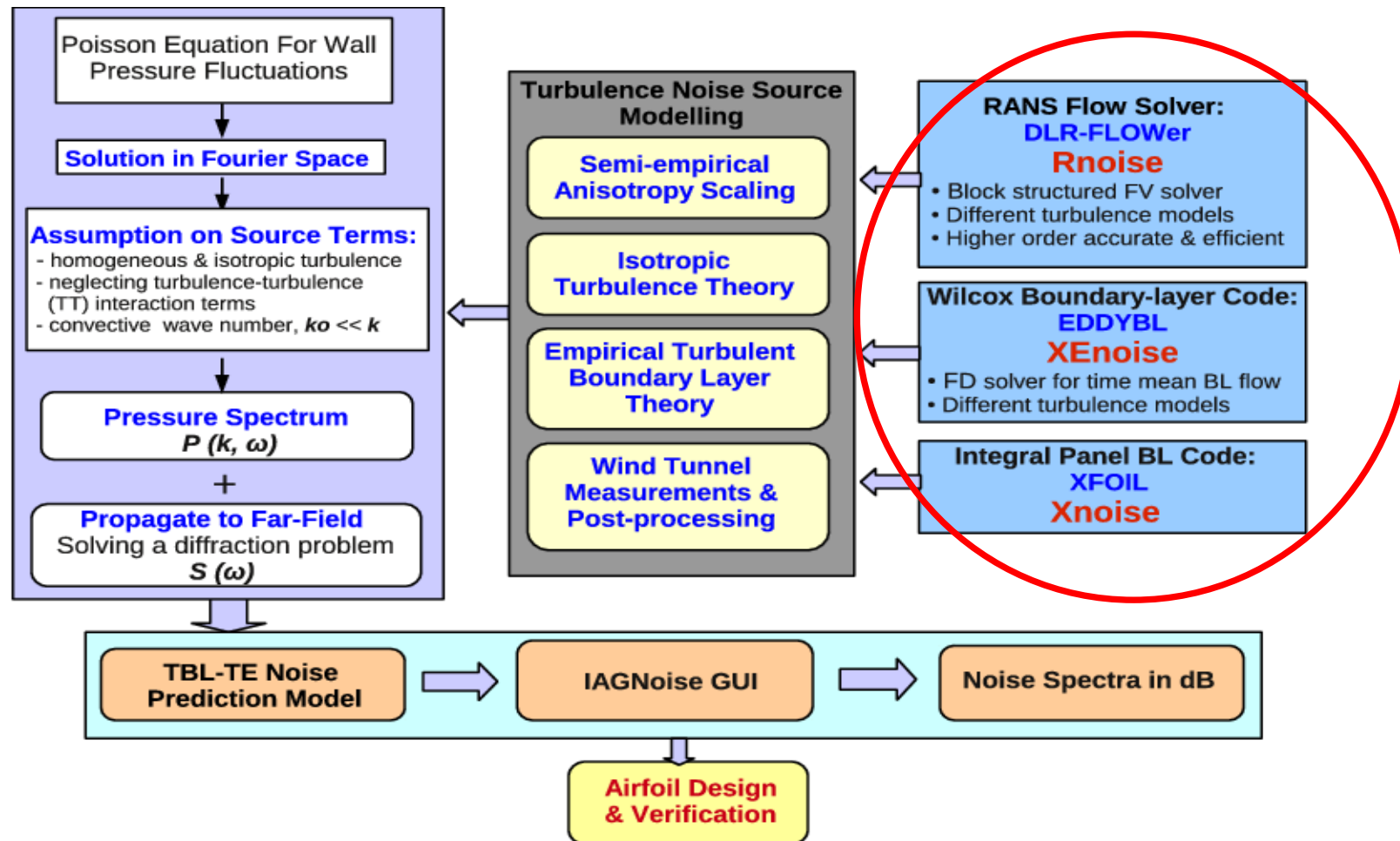


Wagner et al.: Wind Turbine Noise



S. Oerlemans (NLR)

Noise Prediction & Reduction Airfoil Self Noise Prediction Schemes

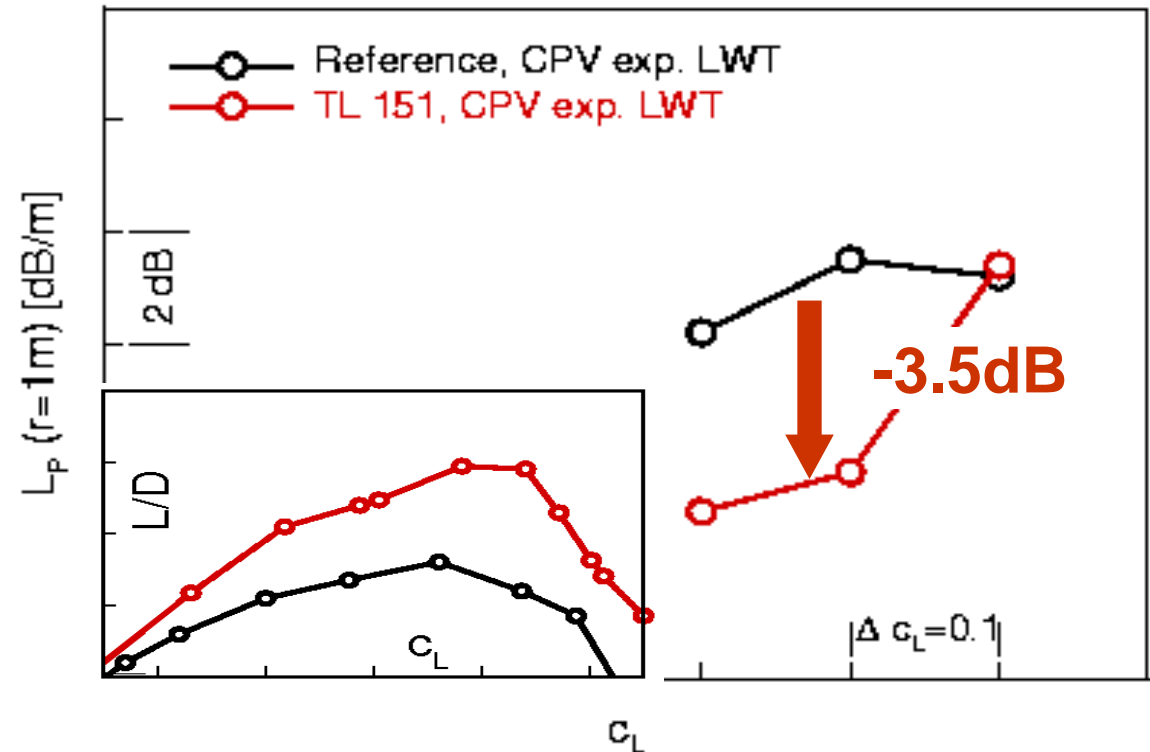


Passive Noise Reduction Aeroacoustic Airfoil Design

GE Reference Turbine
(2.3 MW, $D=94\text{m}$, Wieringermeer, NL)



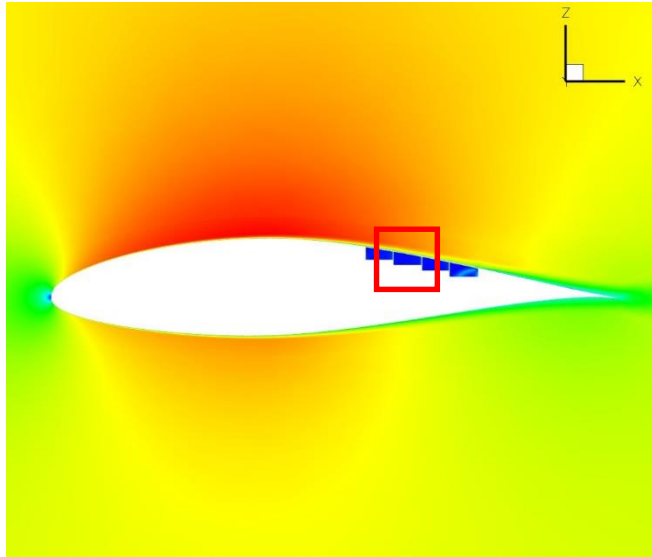
Achieved Gain
(Wind Tunnel Results)



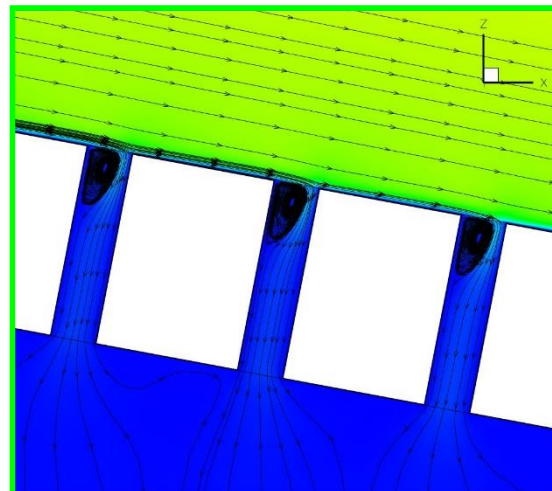
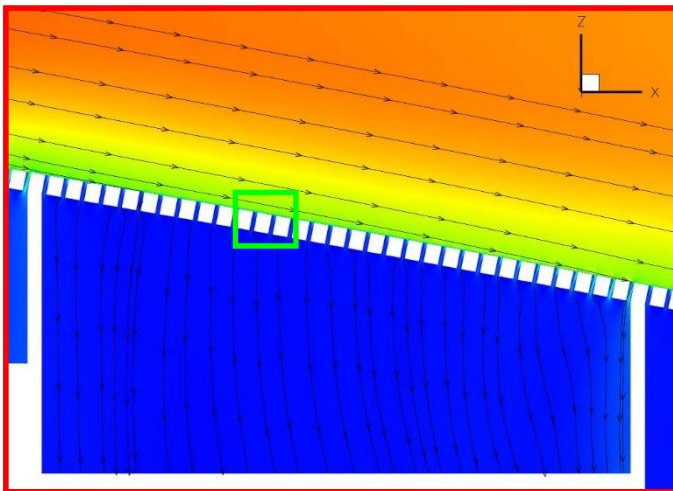
- Reduction of trailing-edge noise
- Increase of performance L/D

Th. Lutz: AIAA Journal, 2007

Noise Reduction by AFC



- Hybrid Mesh for NACA64-418 airfoil with 4 suction chambers on upper side
- Each 0.25mm hole is resolved with 10 points
- 320.000 points, $y^+=1$, BL resolved with 50 points
- Resolving the porous plate delivers a more reliable suction velocity distribution
- Pressure losses at porous plate are considered



TAU simulation performed:

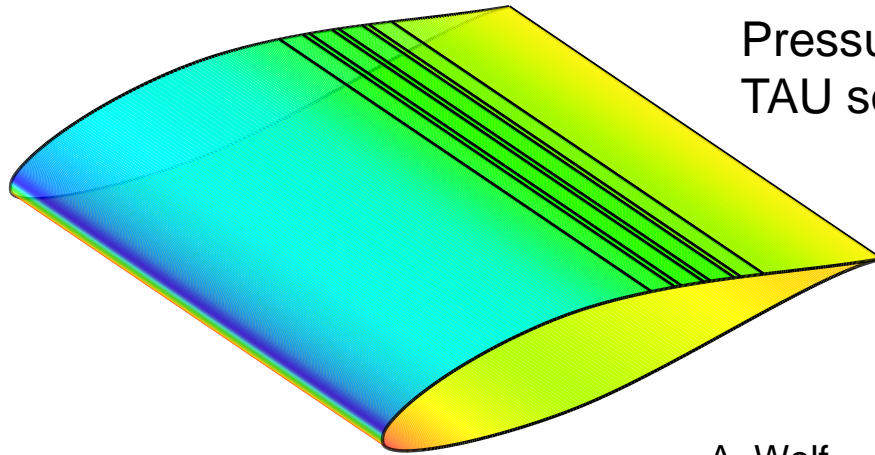
$Ma=0.206$

$Re=0.6$

TM: Wilcox $k-\omega$

A. Wolf

Noise Reduction by AFC CFD vs. Wind Tunnel Experiment

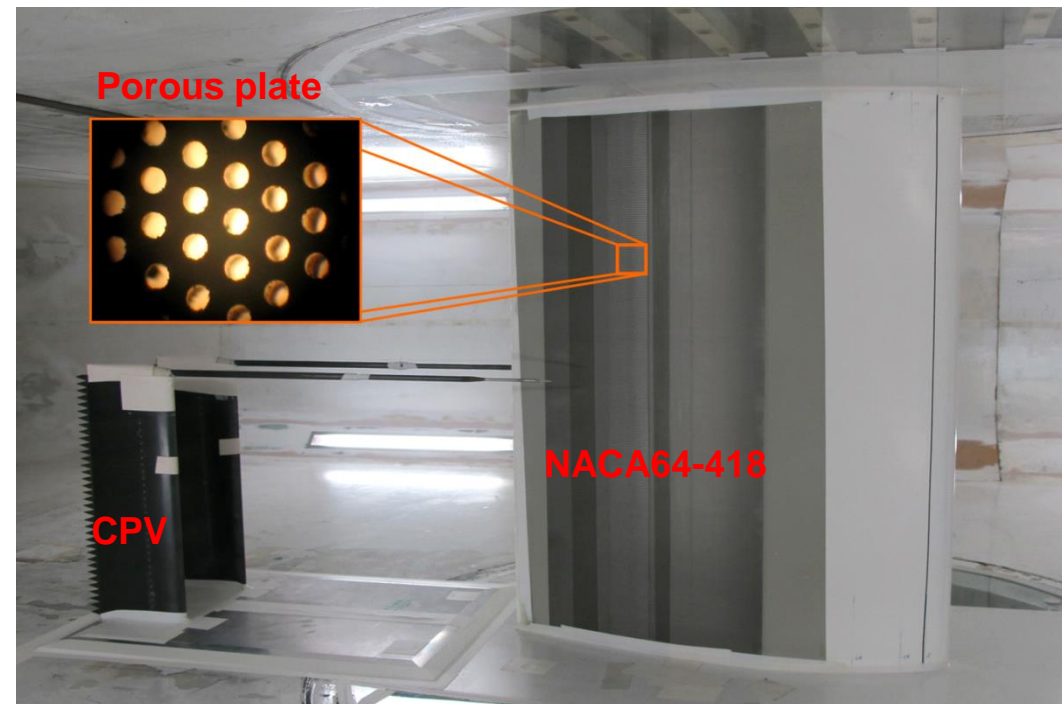


Pressures surface plot of NACA64-418
TAU solution

A. Wolf

NACA64-418 Wind Tunnel Model with
suction device + CPV System for
acoustic measurements

Magnified picture: Porous plate of
suction device, porosity: 25%, hole
diameter: 0.25mm



Noise Reduction by AFC CFD vs. Wind Tunnel Experiment

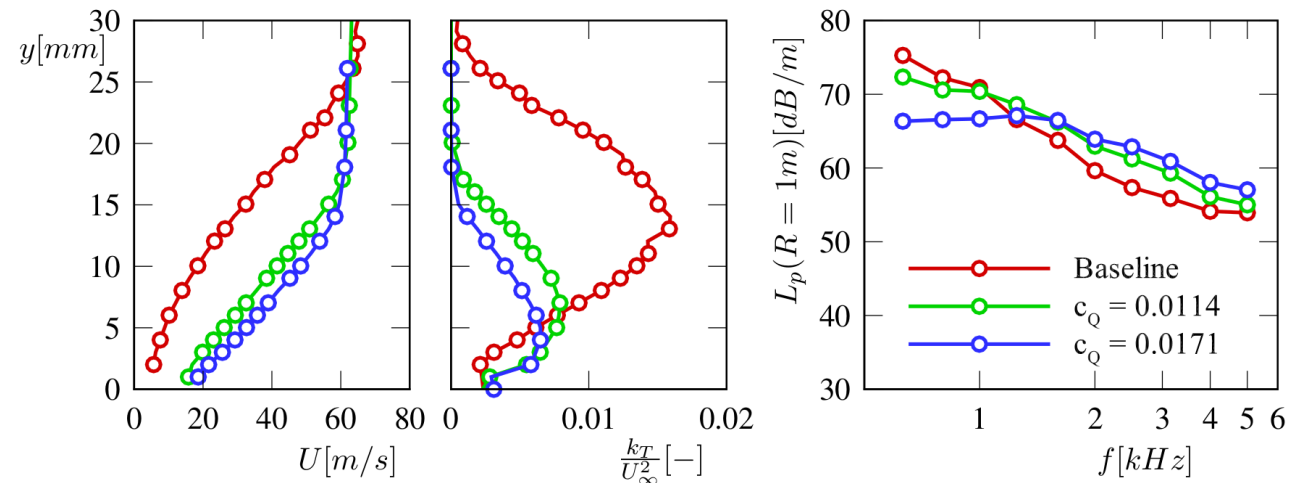
Distributed Suction: Impact of mass flow coefficient

Experiment:

NACA 64-418

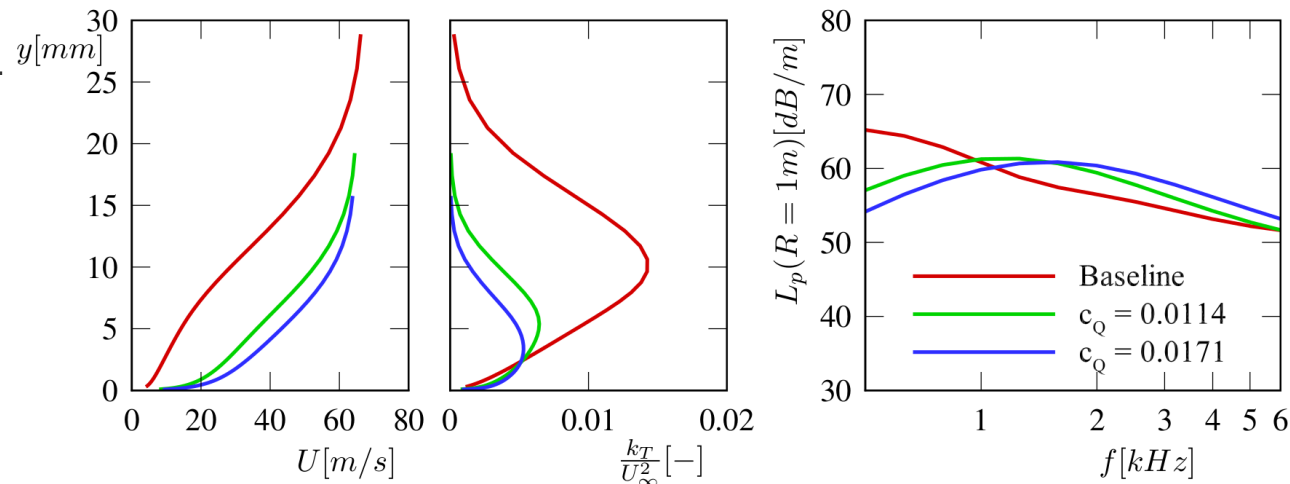
$Re=2.5e6$, $\alpha=6^\circ$

$x_{tr}/c=0.05$



Prediction (Rnoise):

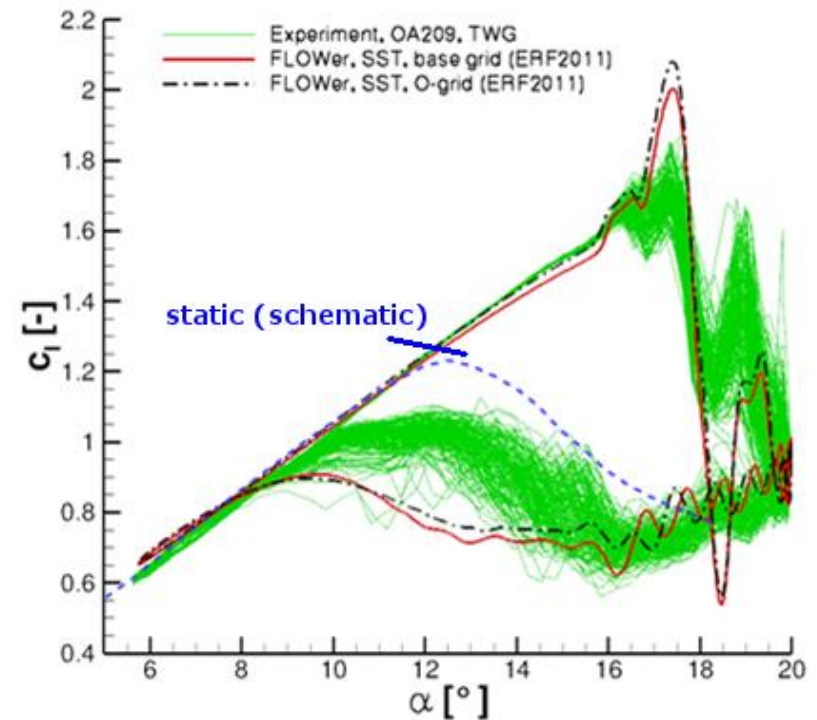
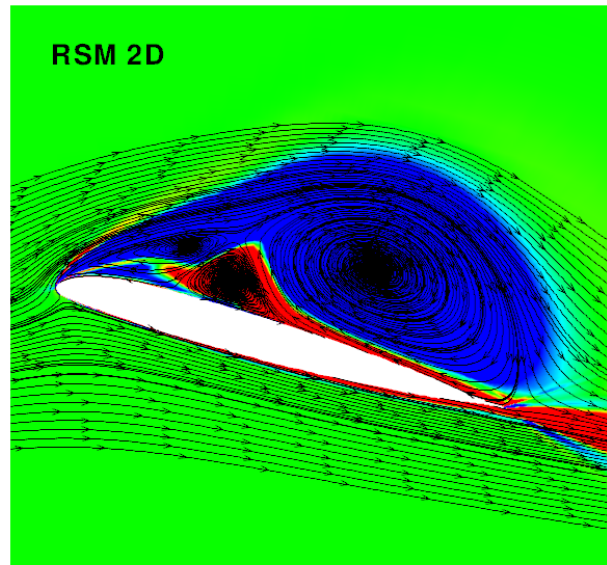
$$c_Q = \frac{\dot{m}_s}{\rho_\infty \cdot U_\infty \cdot A_s}$$



A. Wolf

Dynamic Stall Observations

- Enhanced maximum lift by delay of the stall angle with respect to stationary stall
- Formation of a characteristic large dynamic stall vortex on the suction side
- Strong breakdown of lift and pitching moment due to the downstream convection of the vortex leads to high dynamic loads

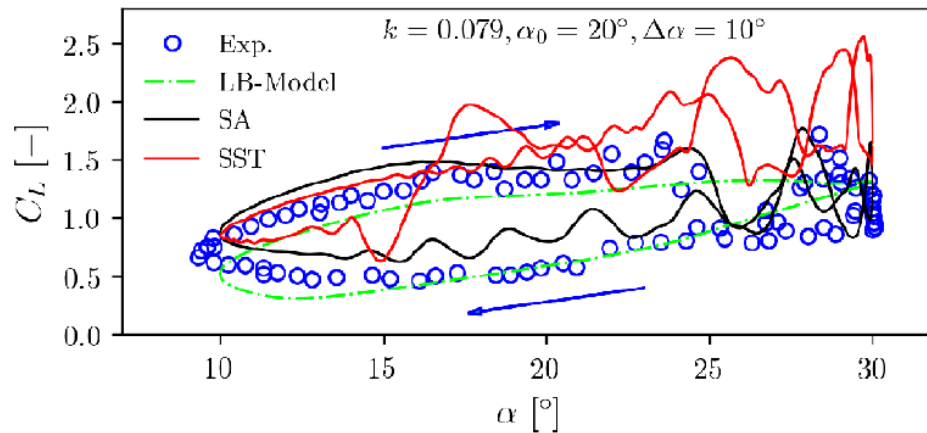


A. Klein

$M = 0.3$, $Re = 1.16e6$, $\alpha_0 = 13^\circ$, $\Delta\alpha = \pm 7^\circ$, $k = 0.05$

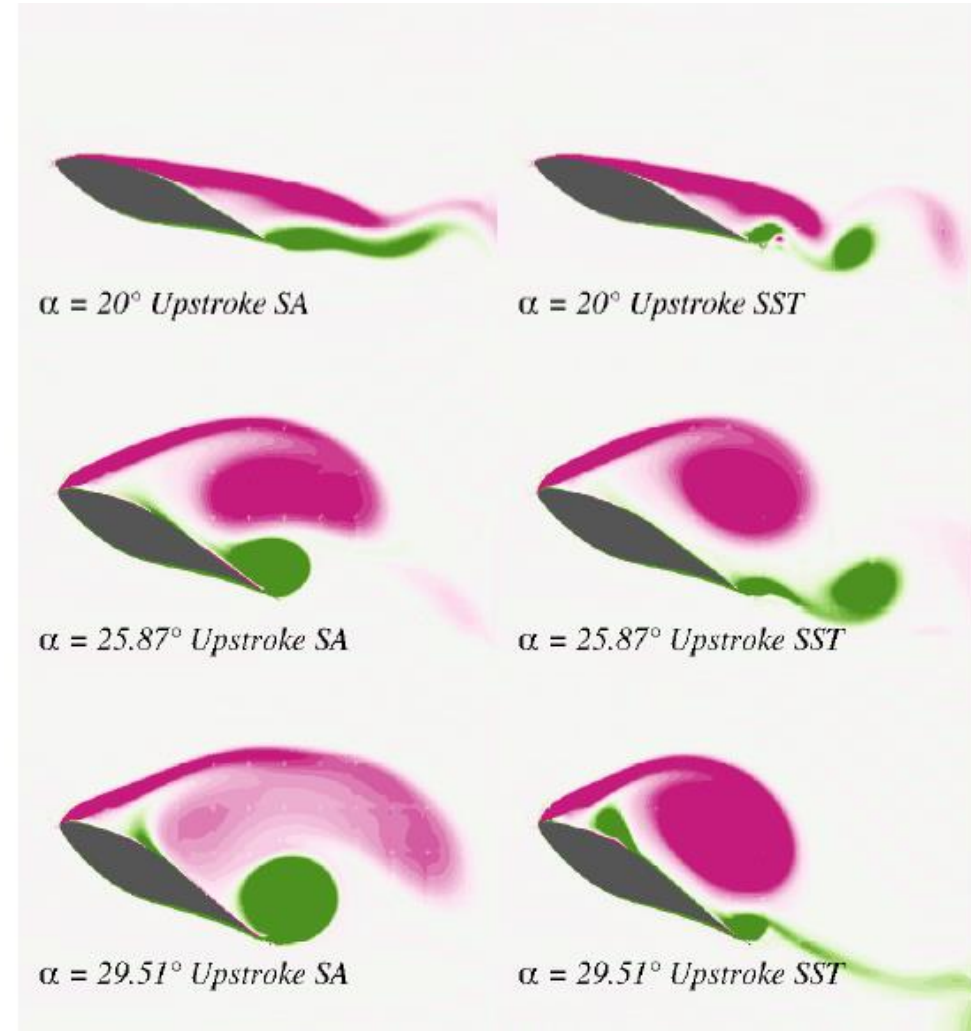
Dynamic Stall Prediction

Dynamic stall

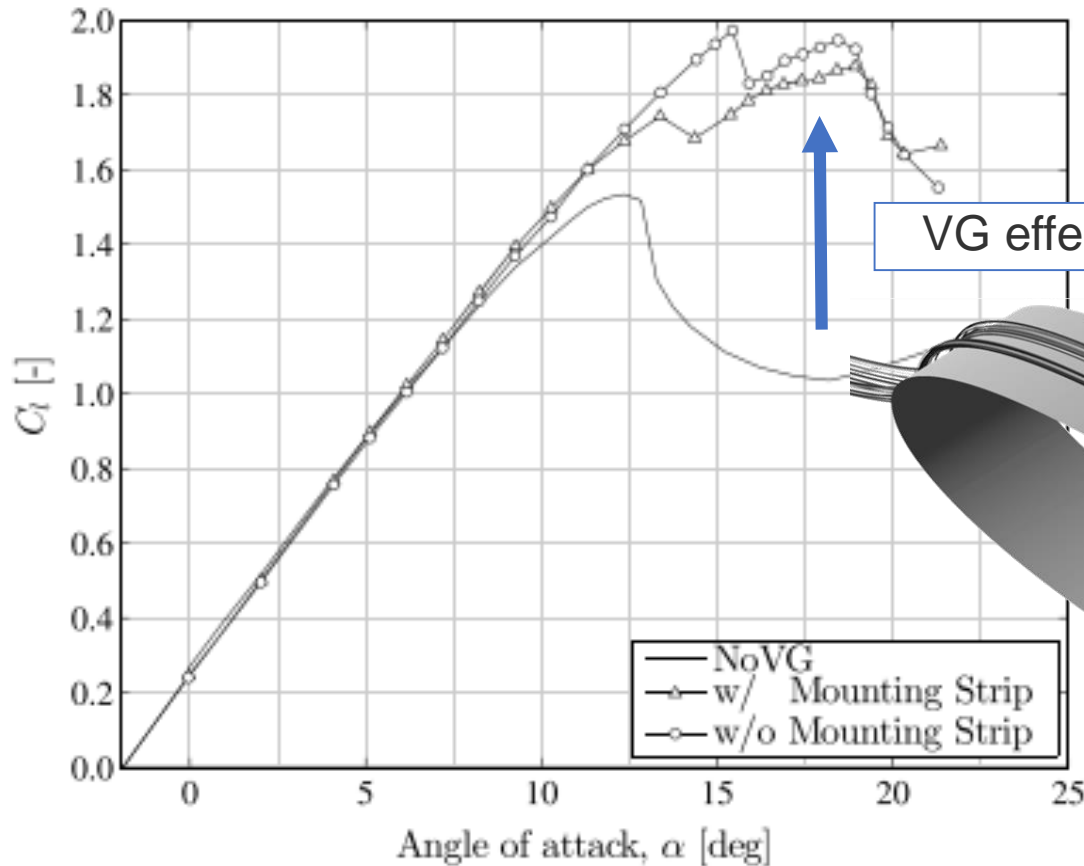


Force and flow field characteristic predictions depend on the employed CFD approaches especially on turbulence model

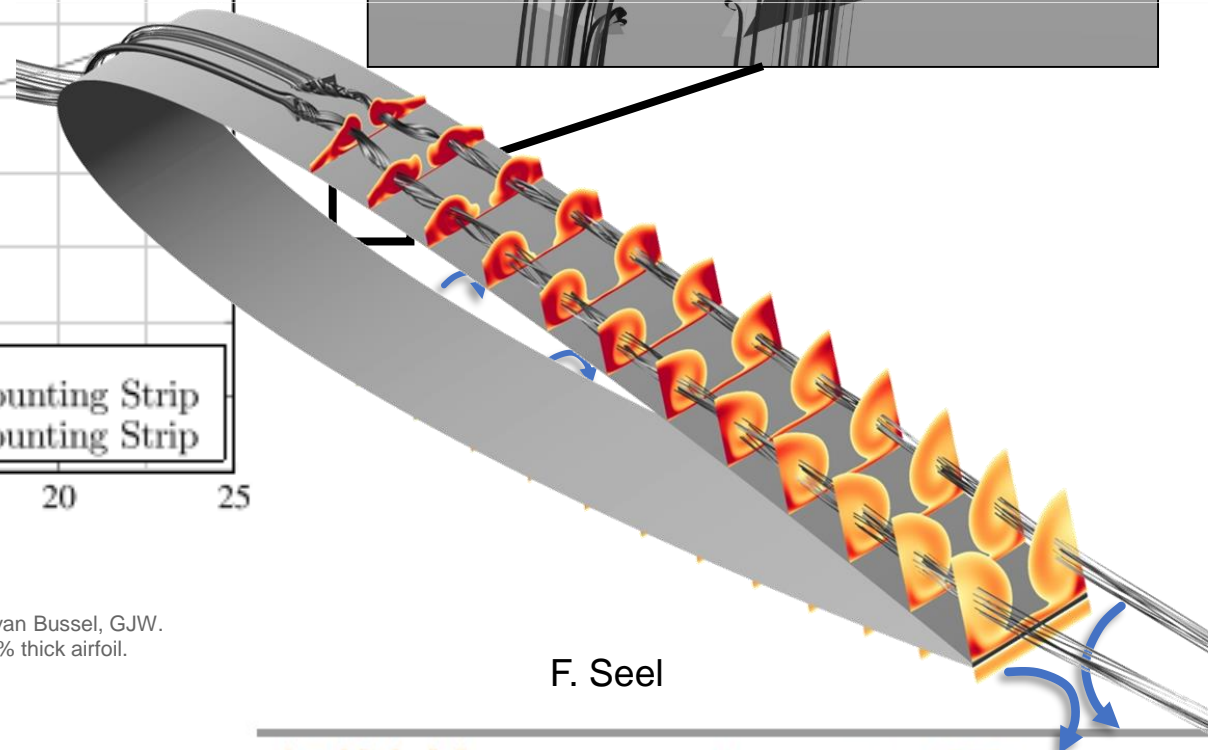
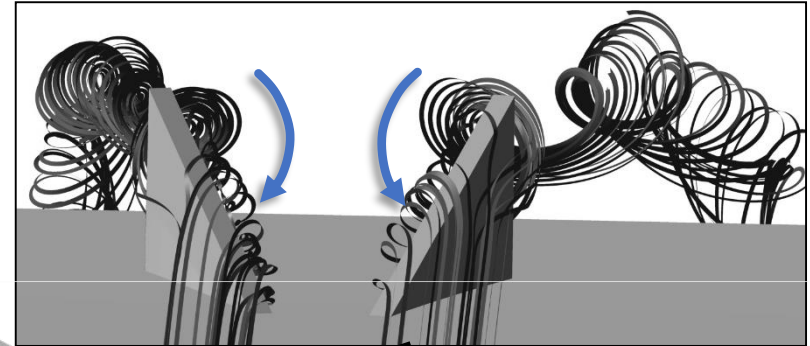
G. Bangga



Vortex Generators Separation Delay & Lift Enhancement



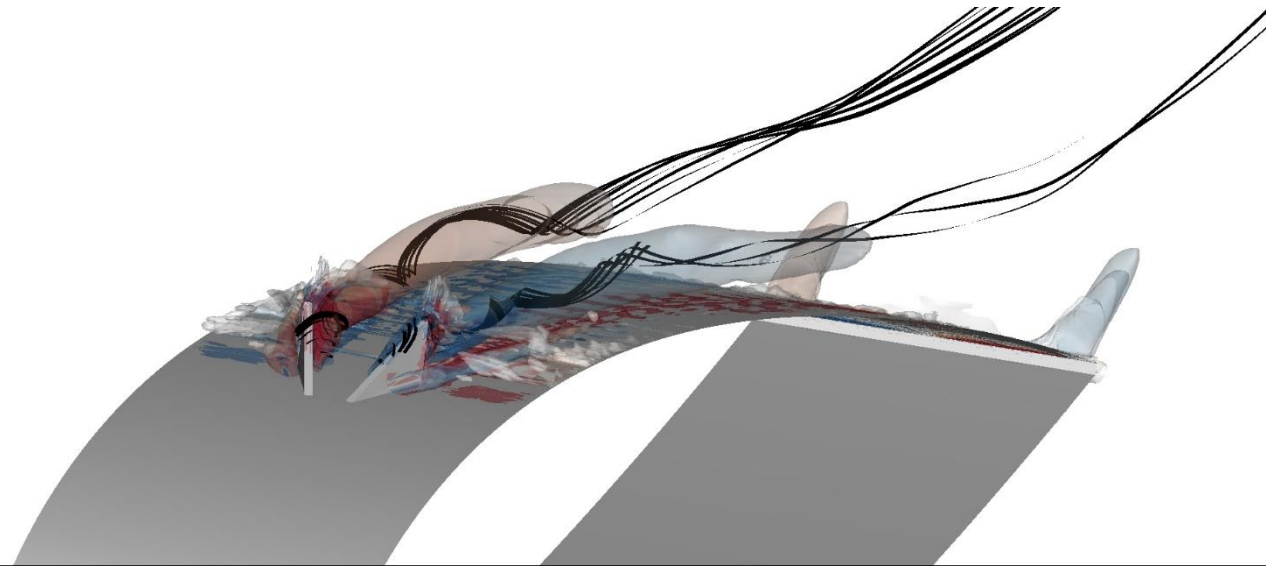
Front view:



F. Seel

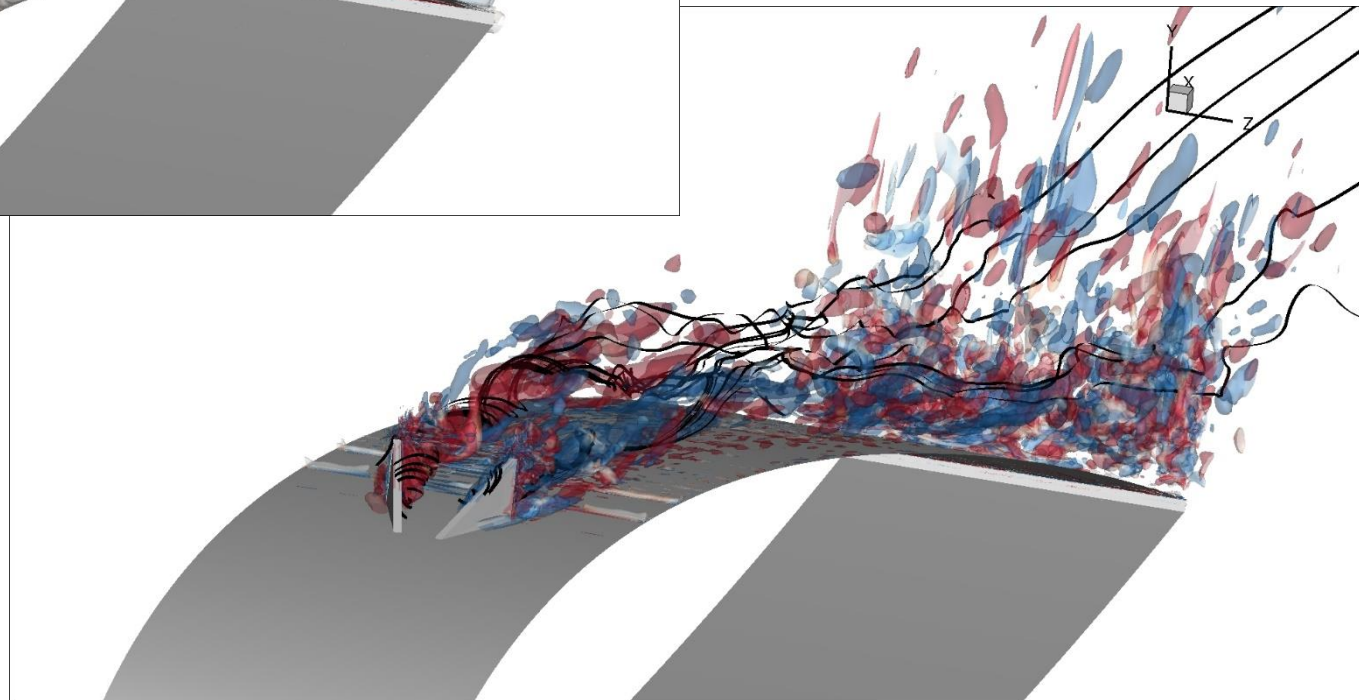
Source: Baldacchino, D, Ferreira, C, De Tavernier, D, Timmer, WA, van Bussel, GJW.
Experimental parameter study for passive vortex generators on a 30% thick airfoil.
Wind Energy. 2018; 21: 745– 765. <https://doi.org/10.1002/we.2191>

Vortex Generators Impact of Turbulence Model on Predicted Mixing



URANS result

Scale resolving iDDDES



F. Seel

Rotor & Wake Flow Physics The Mexico Rotor



DNW LLF



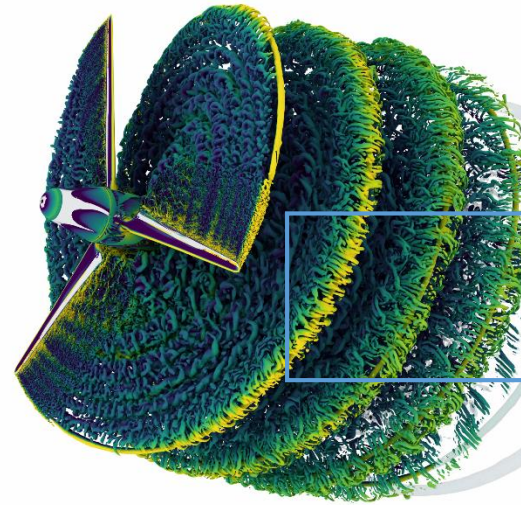
Mexico Rotor

Rotor & Wake Flow Physics Tip Vortex Development

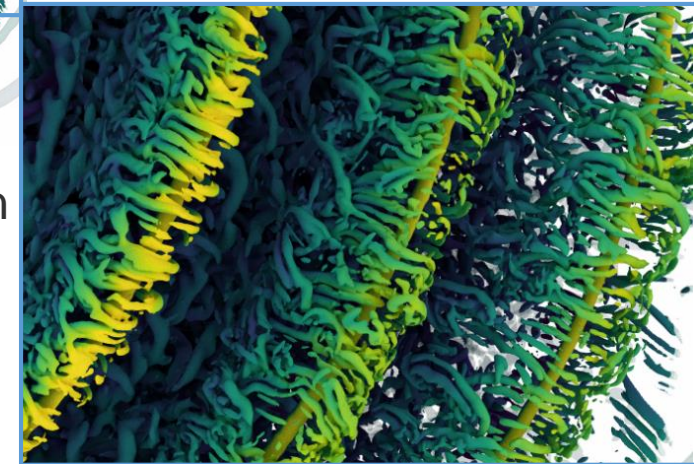
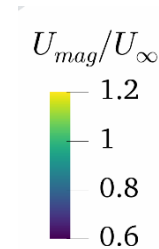
Simulation of the MEXICO model wind turbine at low TSR



Smoke visualization in the experiment



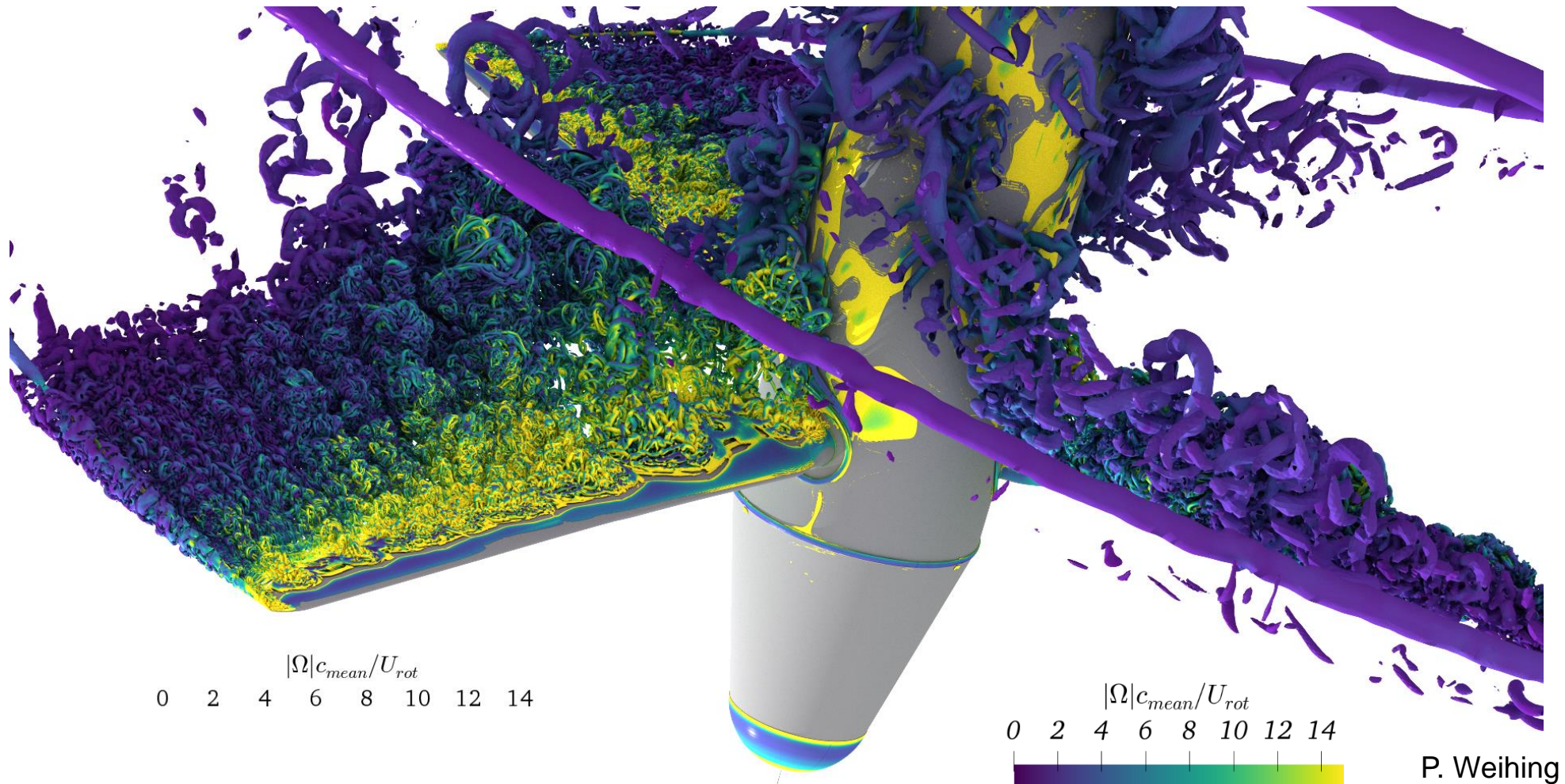
High-fidelity simulation



P. Weiing

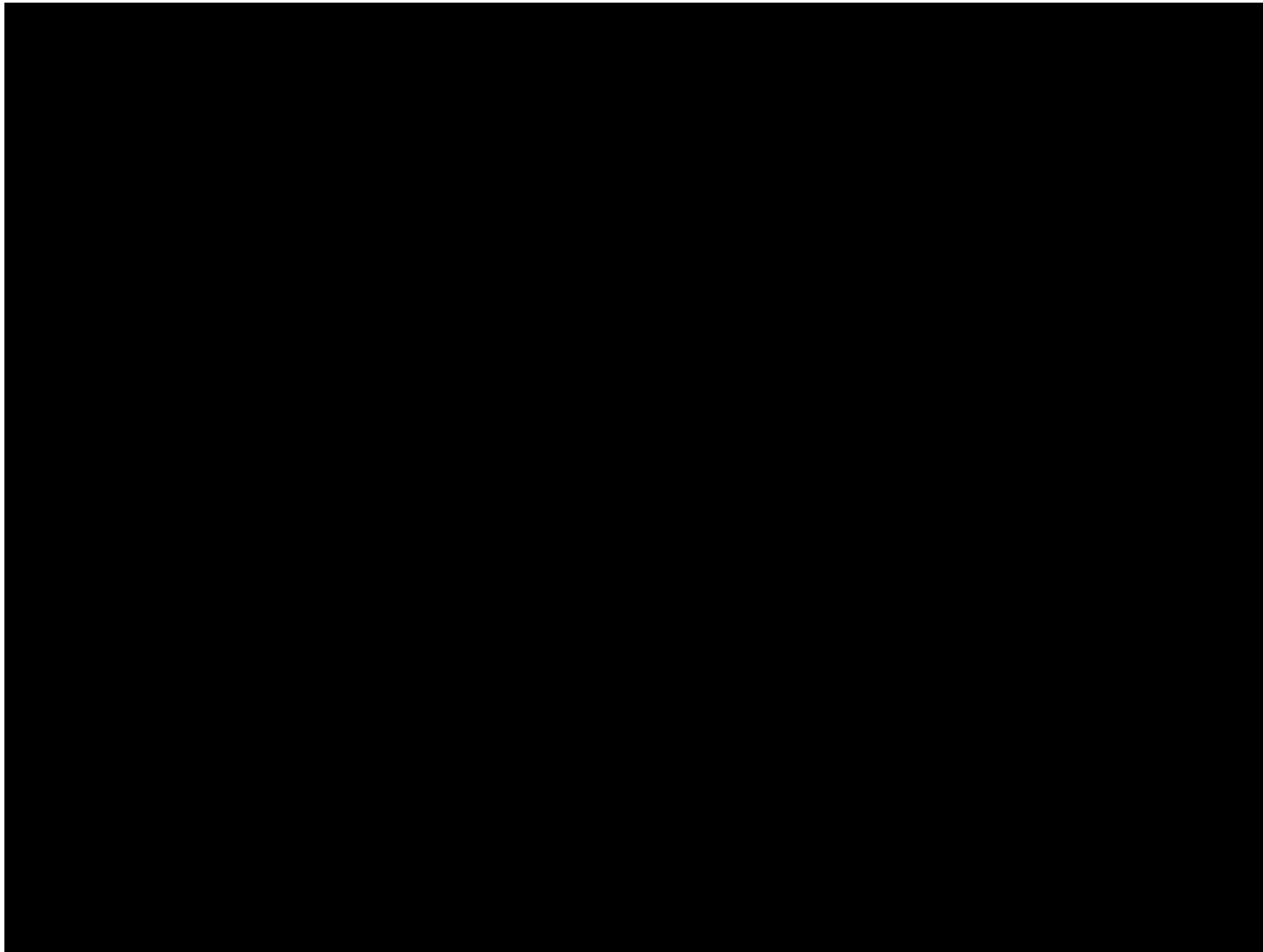
Formation of "turbulent worms"

Rotor & Wake Flow Physics BDES Result

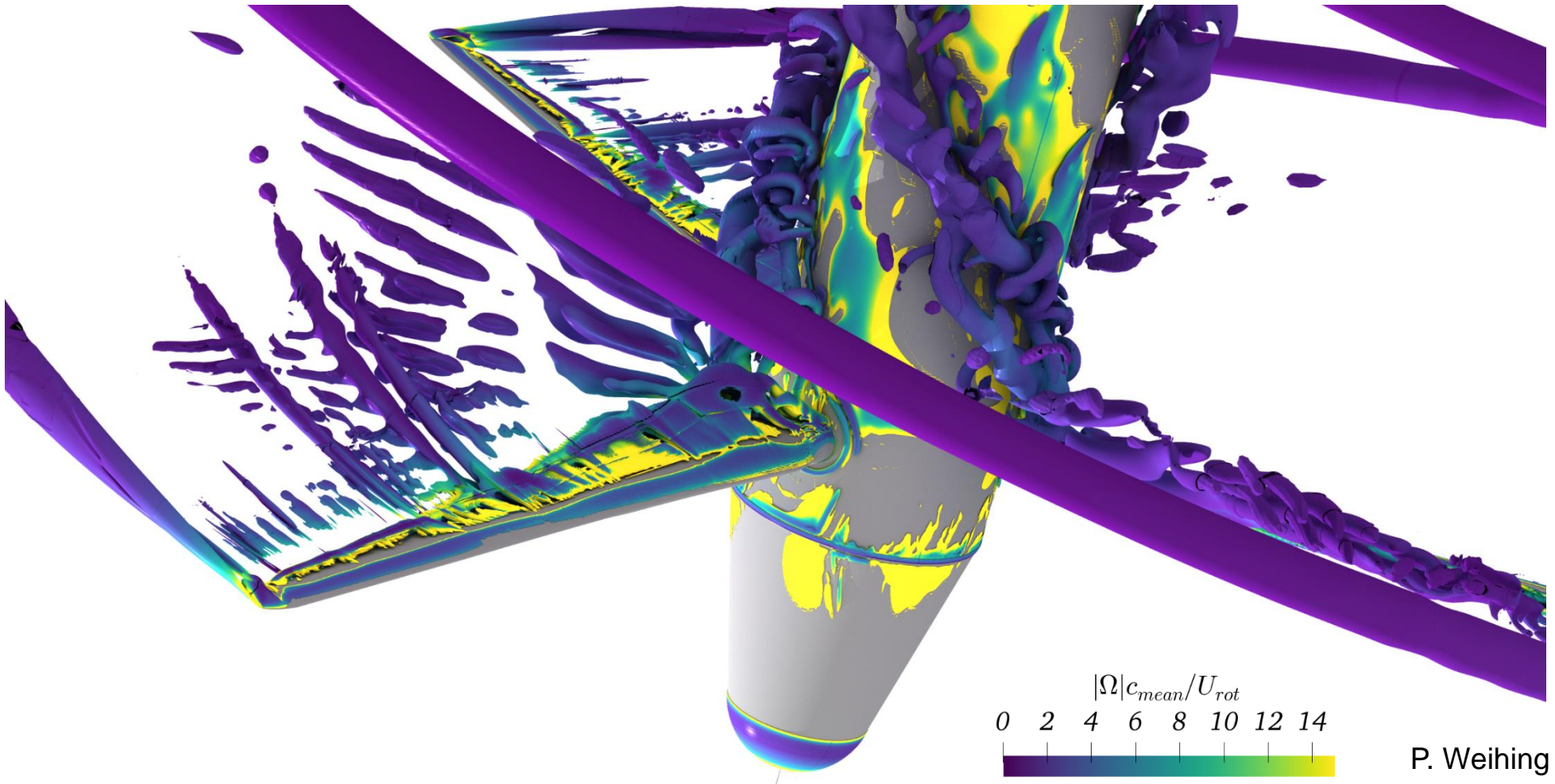


P. Weihsing

Rotor & Wake Flow Physics BDES Result - Video

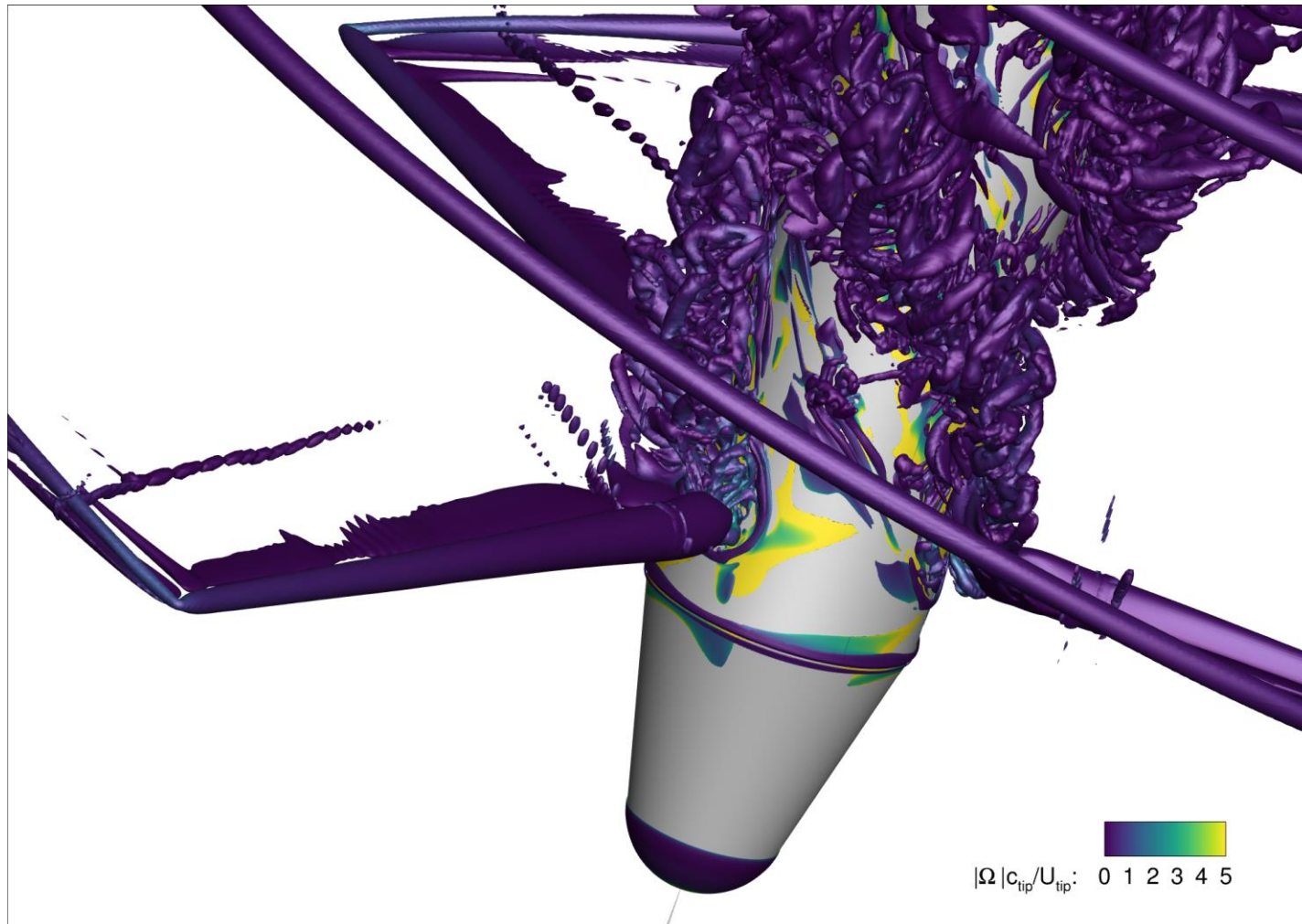


Rotor & Wake Flow Physics URANS Result



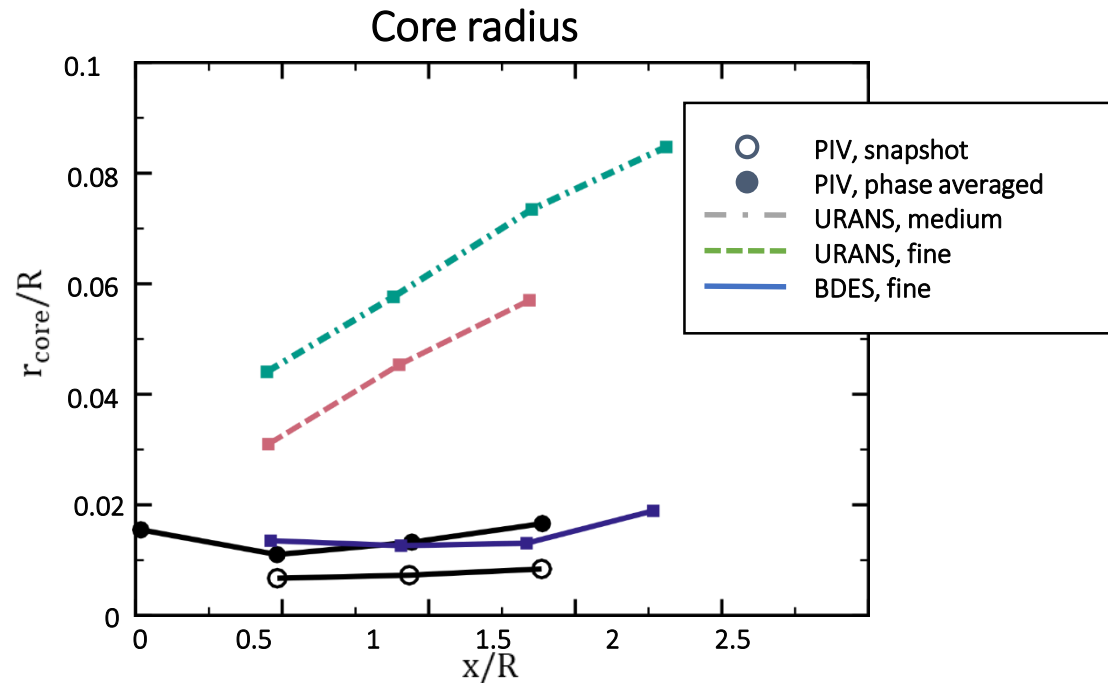
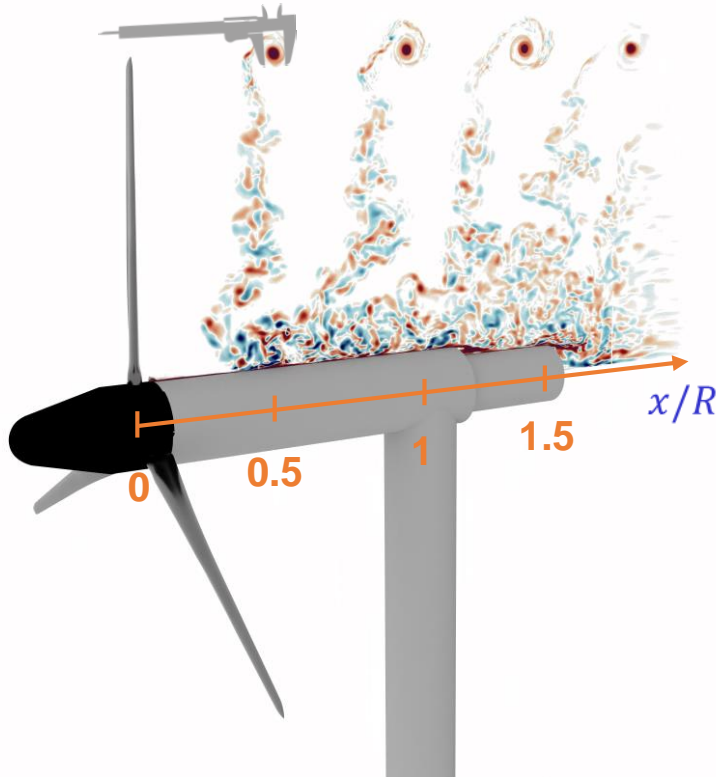
P. Weihsing

Rotor & Wake Flow Physics Actuator Line Result

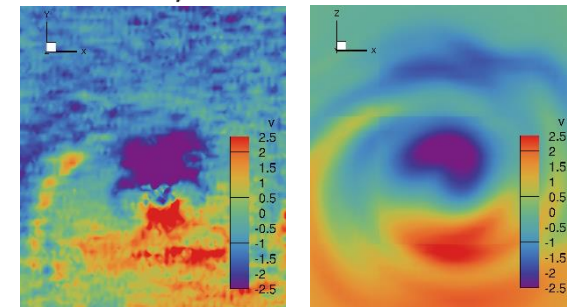


M. Cormier

Rotor & Wake Flow Physics Development of Tip Vortex Radius



- Excessive vortex diffusion for URANS
- Highly improved vortex preservation with DES



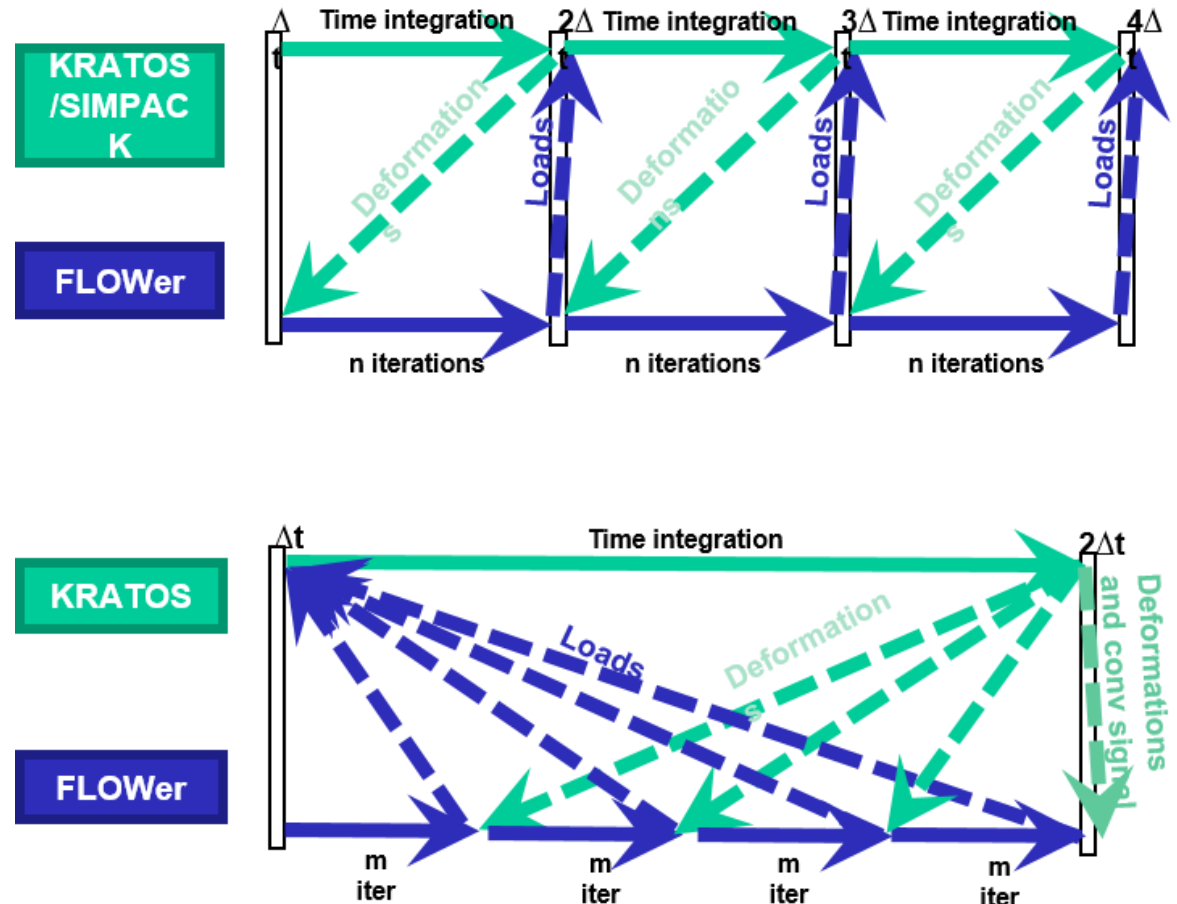
P. Weiing

Aeroelasticity Fluid Structure Coupling Schemes

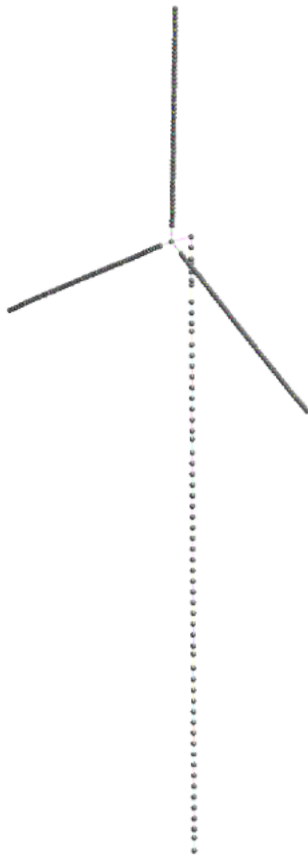
FLOWer is coupled to two different structural solvers:

- MBD with beam elements in *SIMPAC*
- FEM with beam or full FE shell elements in *Kratos*

Both, implicit and explicit coupling schemes are possible



Aeroelasticity Structural Models in Kratos



Beam - Model

$\sim 10^3$ DOFs

- Easy to model
- Cross-Sectional properties
- Fast simulation
- Euler-Bernoulli & Timoshenko
- Non-linear
- 133 nodes for the blade



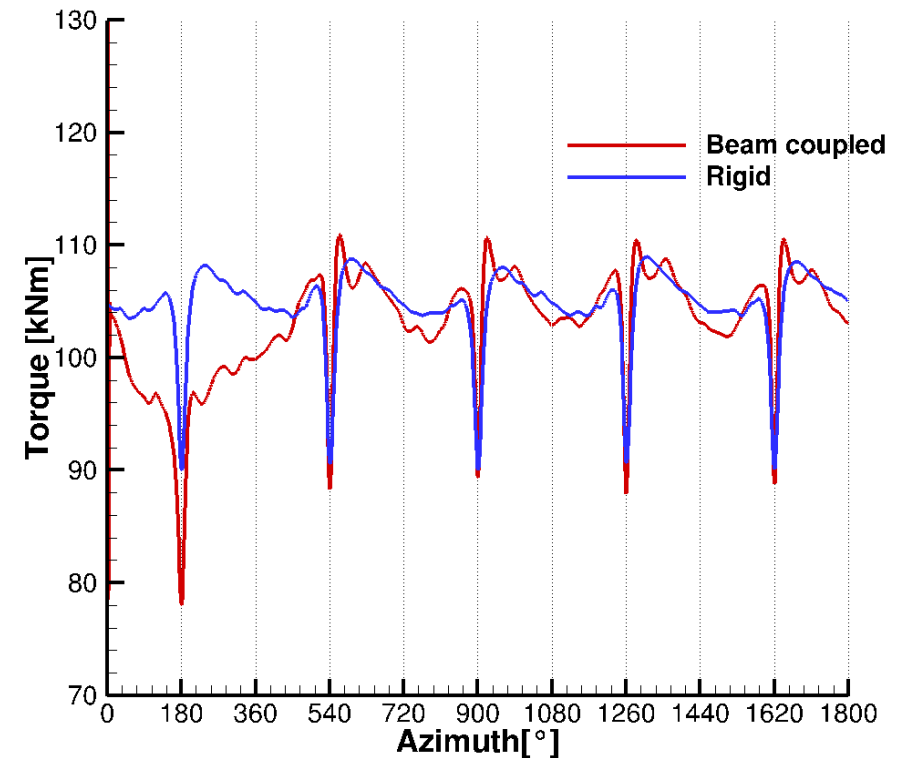
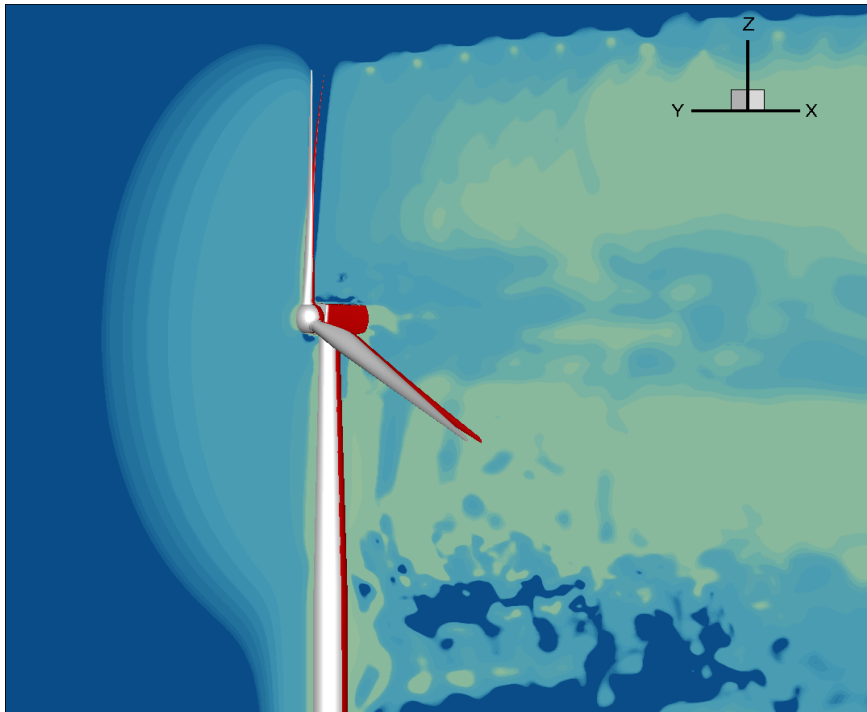
Shell - Model

$\sim 10^6$ DOFs

- Accurate description of local effects
- “Real” material properties
- Non-linear
- Geometry of wet surface available for FSI
- 19215 nodes for the blade

G. Guma

Aeroelasticity Flexible Turbine in Uniform Inflow

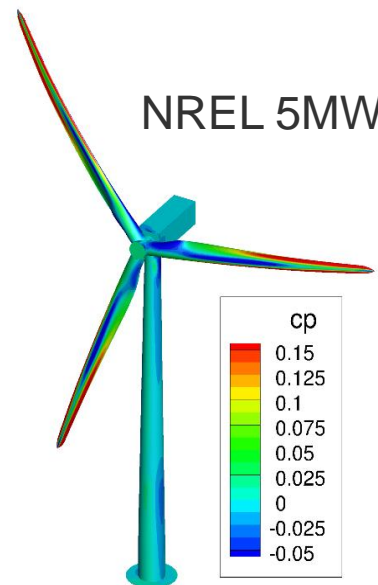
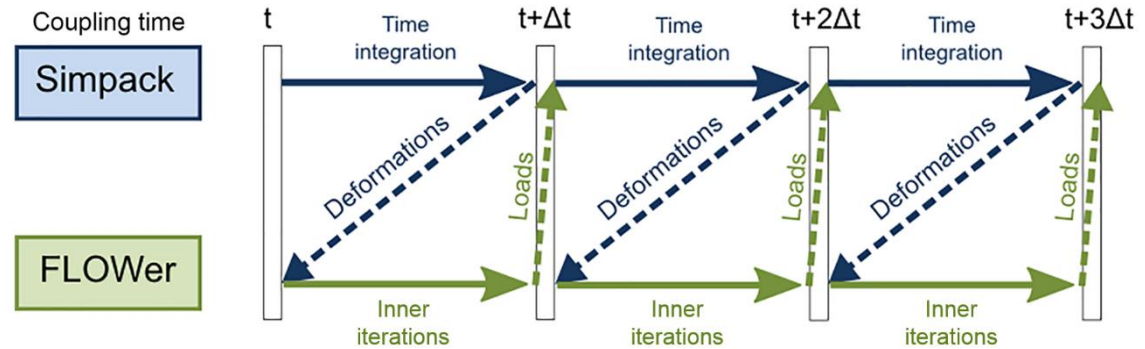


- Loads relaxation at the beginning to support convergence
- Power: approx. -0.5% with flexibility
- Max tip deformation: 6.6% radius (vs 5.8% if only a blade model)
- Tower top contributes with ~0.6% radius

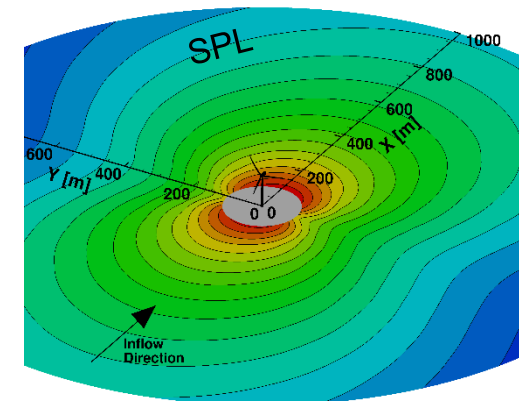
G. Guma

Low Frequency Noise CFD – MBS – FW-H Chain

- Explicit fluid-structure coupling
 - Non-linear beam deflection
 - Rigid body motion
 - Variable pitch / rotation speed
- Ffowcs-Williams Hawkins acoustic propagation
 - Use pressure fluctuations on turbine surface
 - Get acoustic pressure time series at discrete observer positions



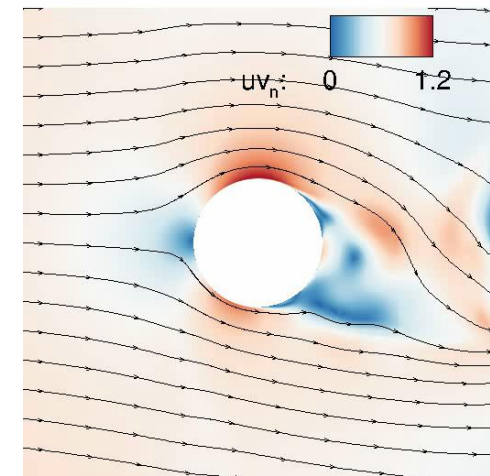
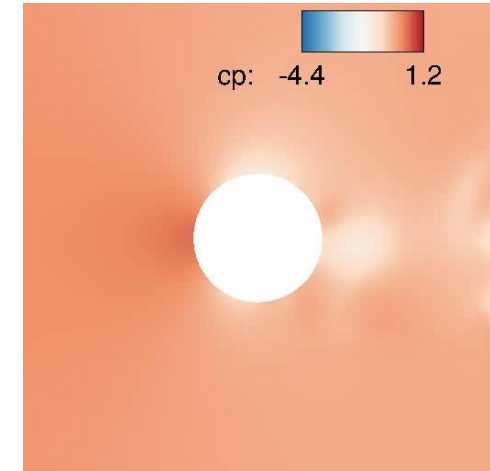
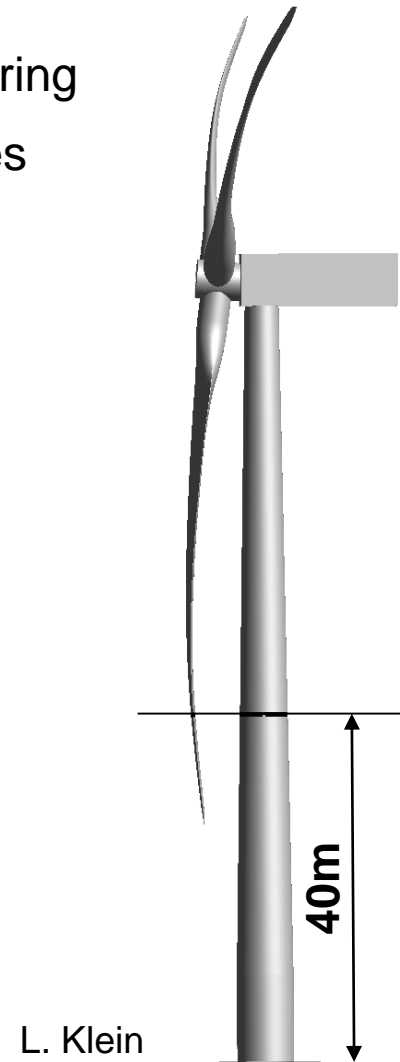
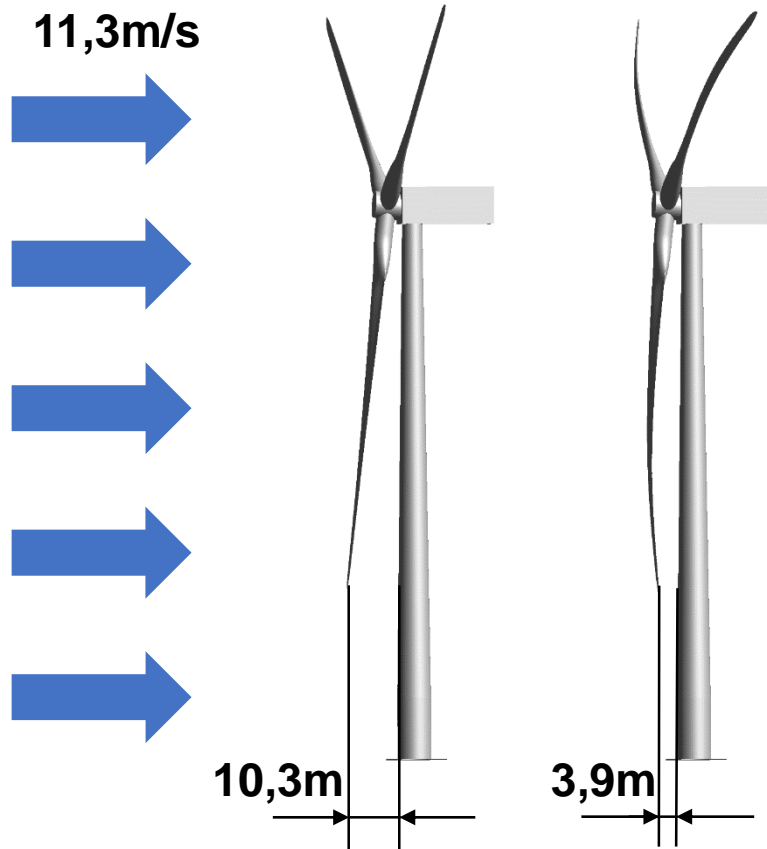
ACCO



L. Klein

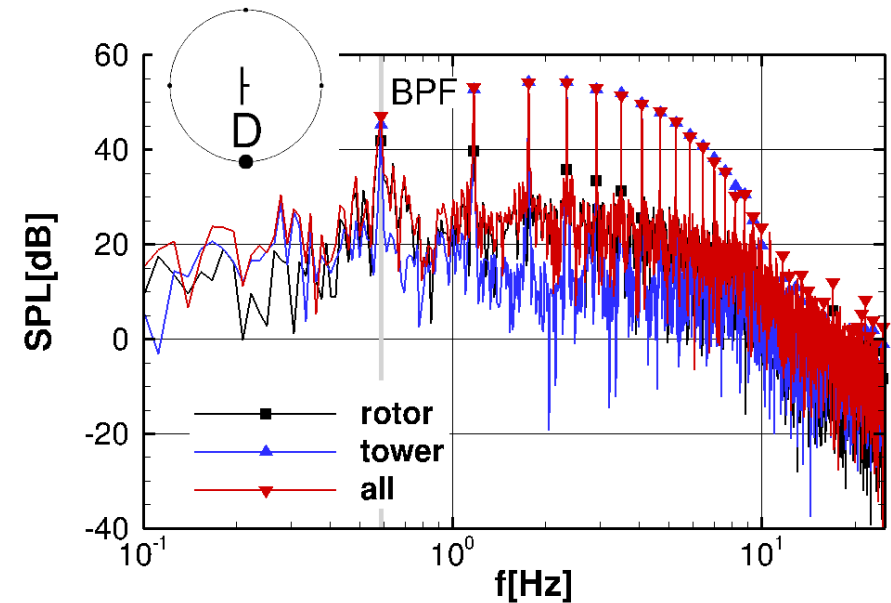
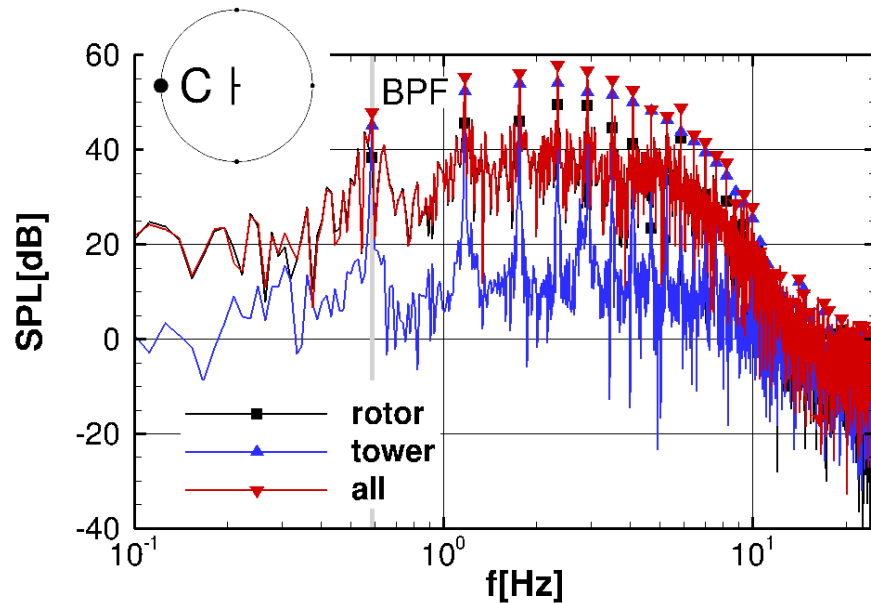
Low Frequency Noise Flexible Turbine in Uniform Inflow

Unsteady viscous flow field during tower passage of flexible blades



Low Frequency Noise Flexible Turbine in Uniform Inflow

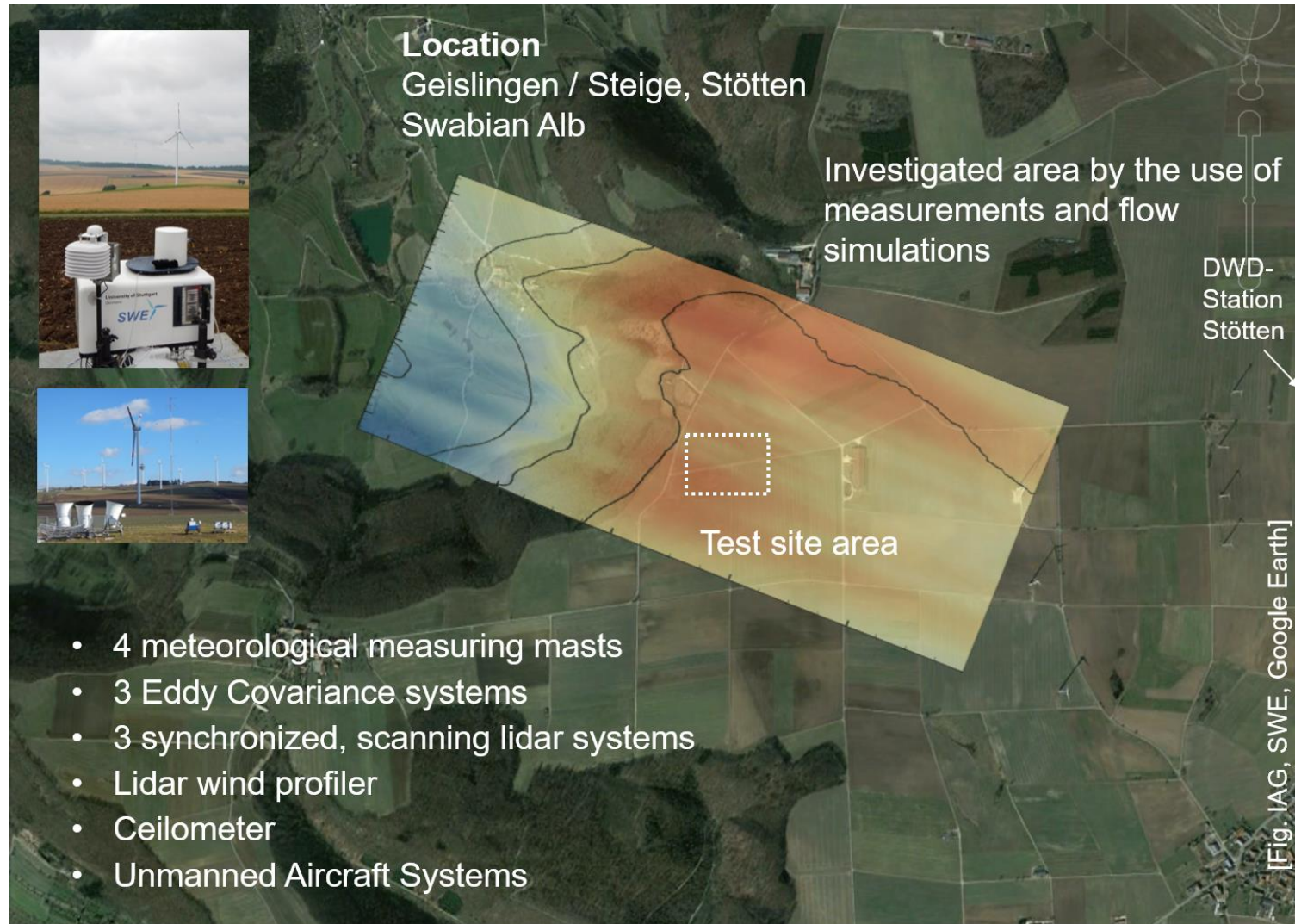
Rigid but statically deformed NREL 5MW turbine at uniform inflow



- **Rotor:** Emits less broadband noise to lateral direction than to for/aft and peaks at BPF harmonics are not very prominent
- **Tower:** Peaks at BPF harmonics have much higher amplitudes and are more prominent
- **Turbine:** Tower emits majority of low frequency sound

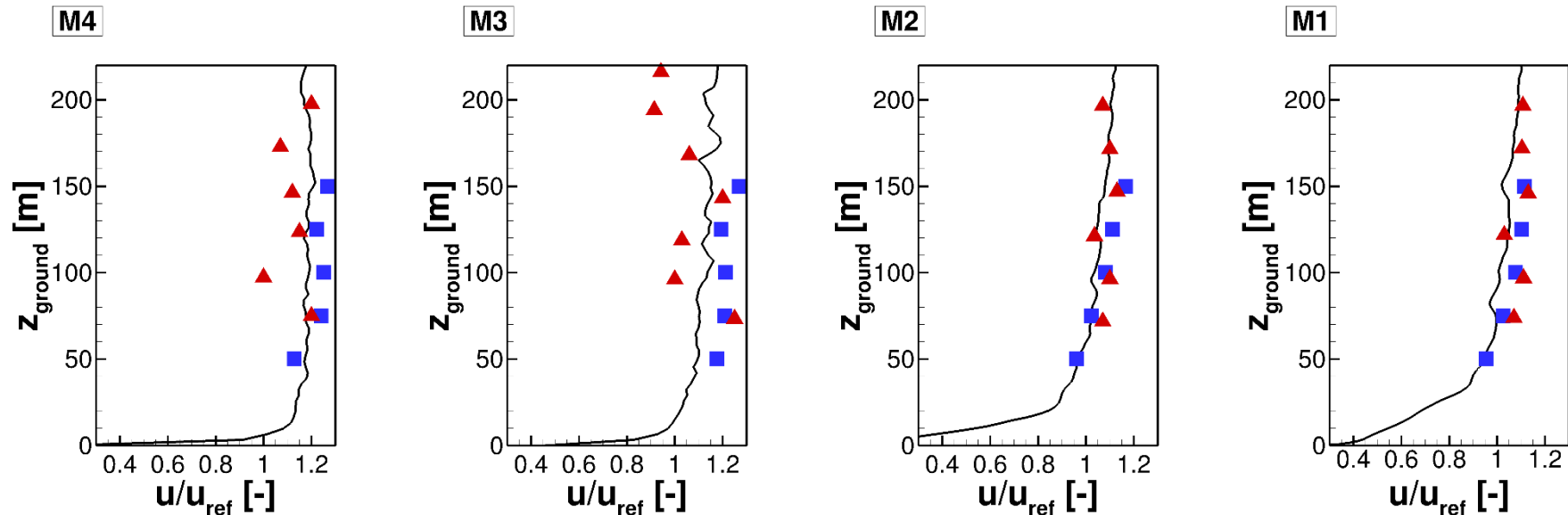
Source: Advanced CFD-MBS coupling to assess low-frequency emissions from wind turbines; L. Klein et al.; Wind Energy Science

Complex Terrain WindForS Test Site (Under Construction)

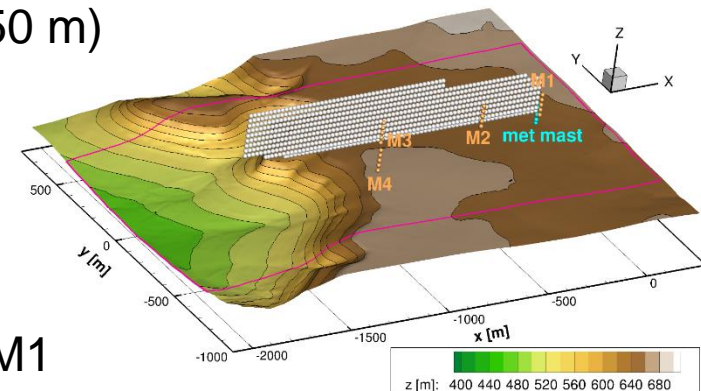


Complex Terrain Terrain Simulation Without Turbine

Predicted vs. Measured Wind Speed Profiles

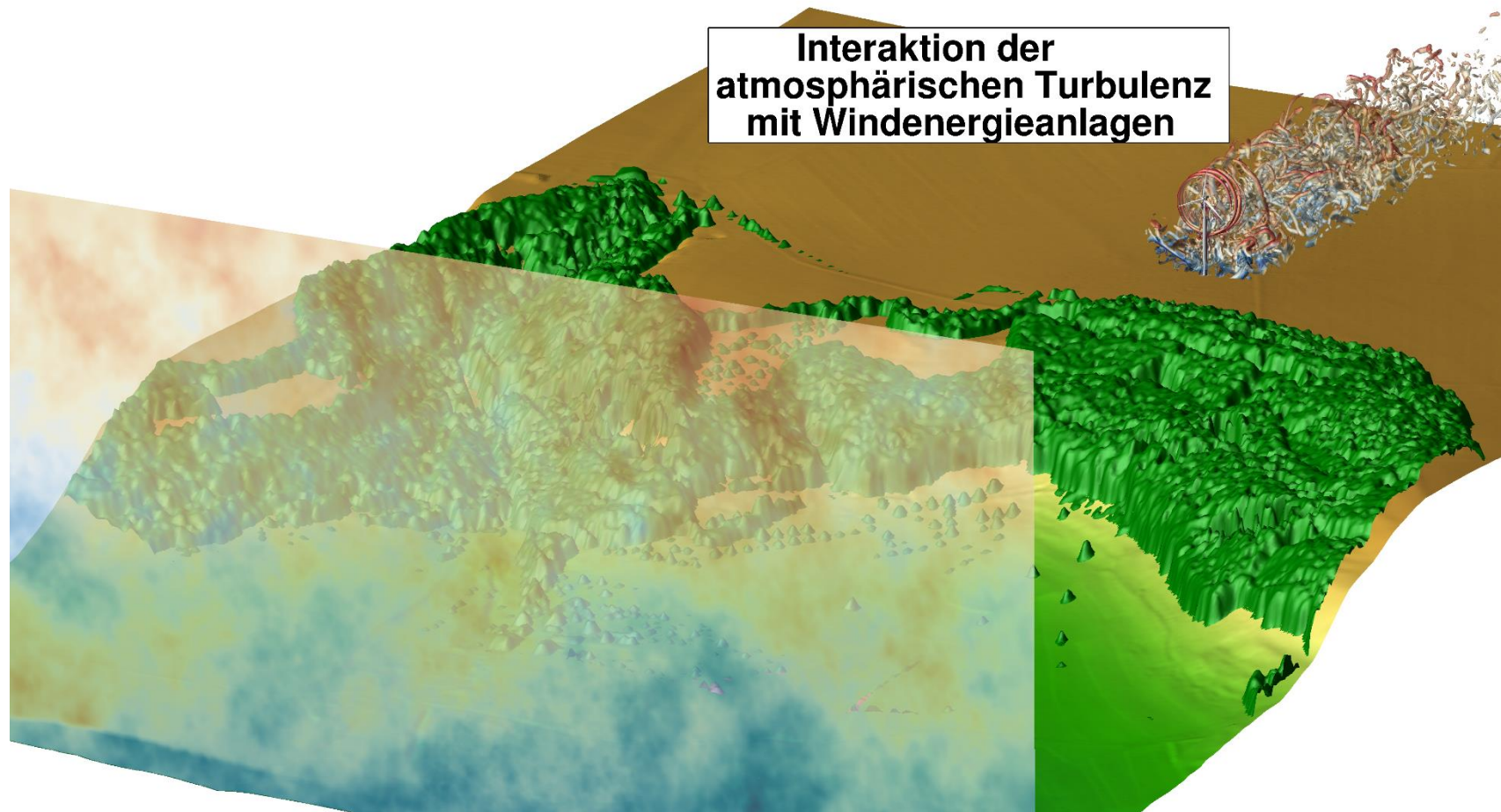


- Increase of velocity at the edge (factor 1.2 in 50 m)
- Recovery of profile downstream
- Lidar: good agreement with simulation
- UAV: deviations at M4 and M3
- UAV: zigzag shape at M4 and M3
- Good agreement between all data at M2 and M1



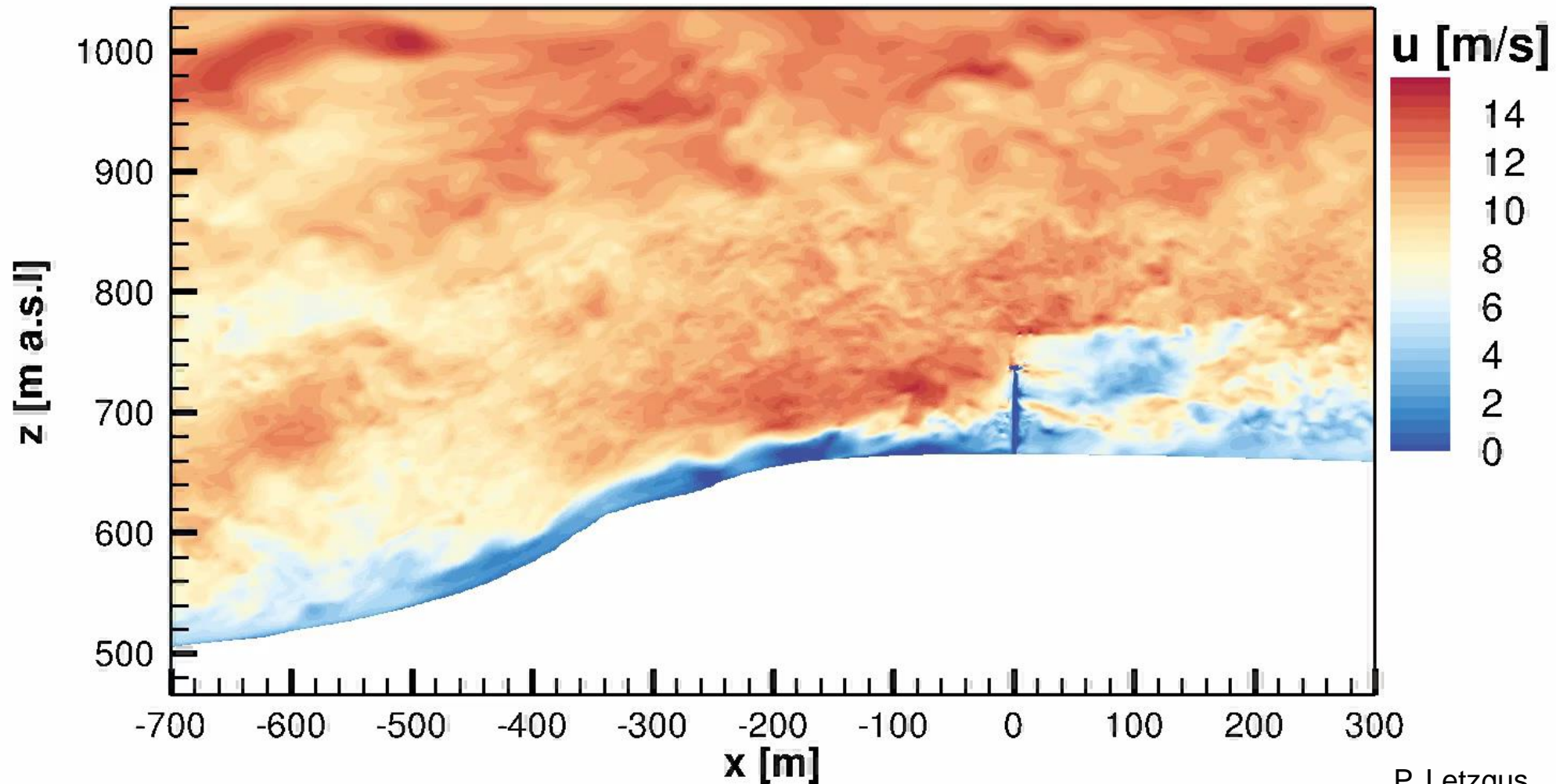
C. Schulz

Complex Terrain Rigid Turbine in Complex Terrain With Forrest

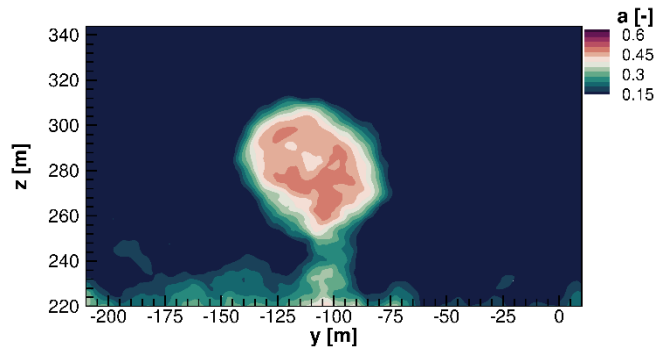


P. Letzgus

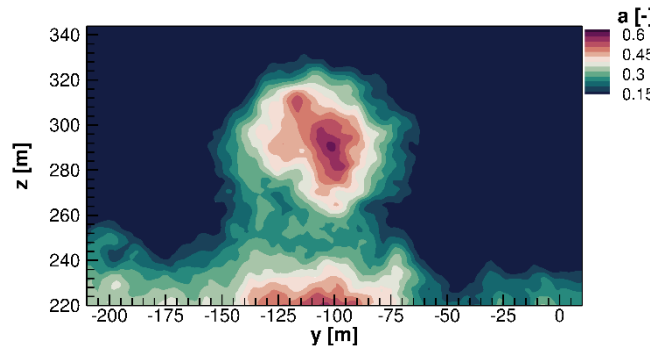
Complex Terrain Rigid Turbine in Complex Terrain With Forrest



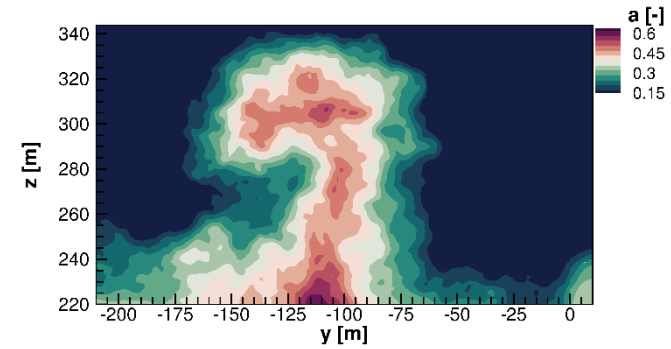
Complex Terrain Impact of Stratification on Turbine Wake



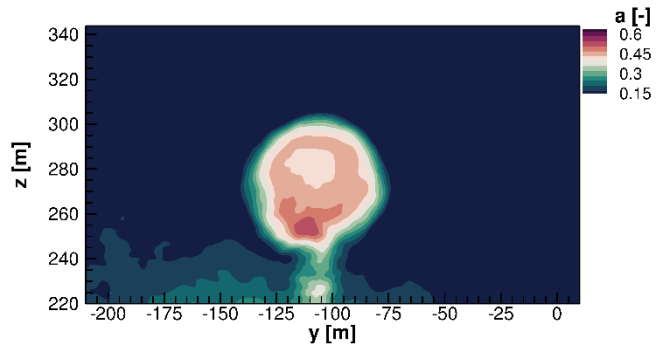
Unstable 2D



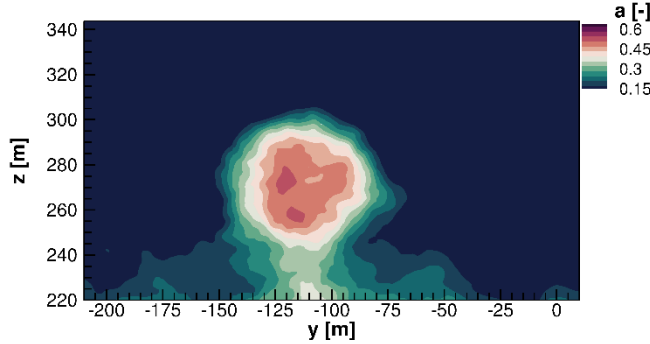
Unstable 4D



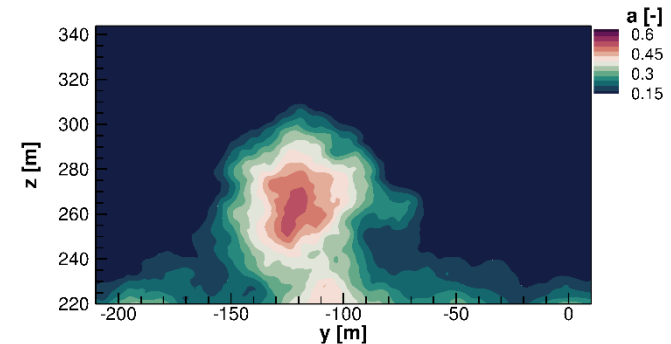
Unstable 6D



Stable 2D



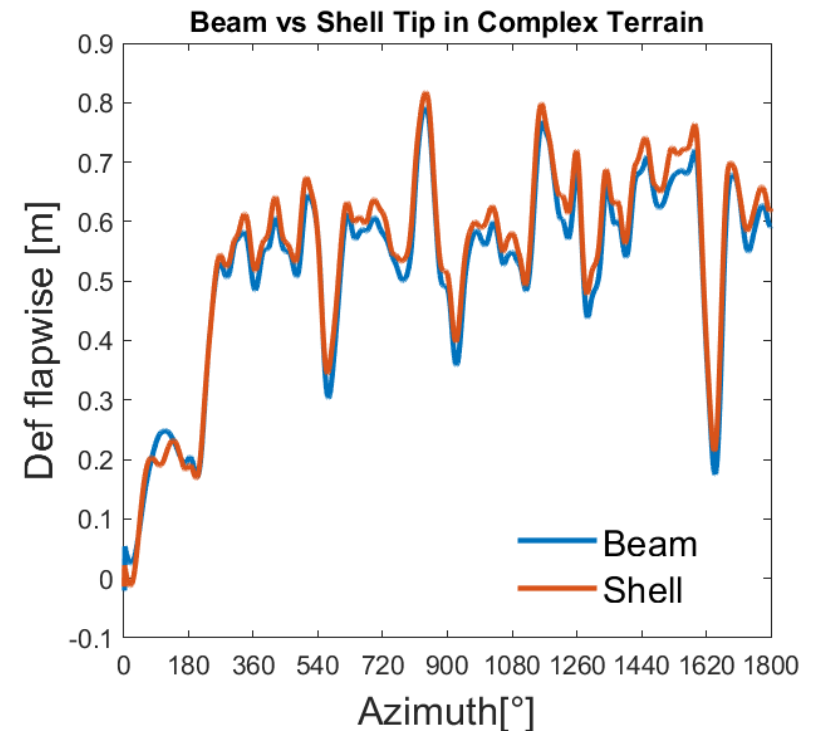
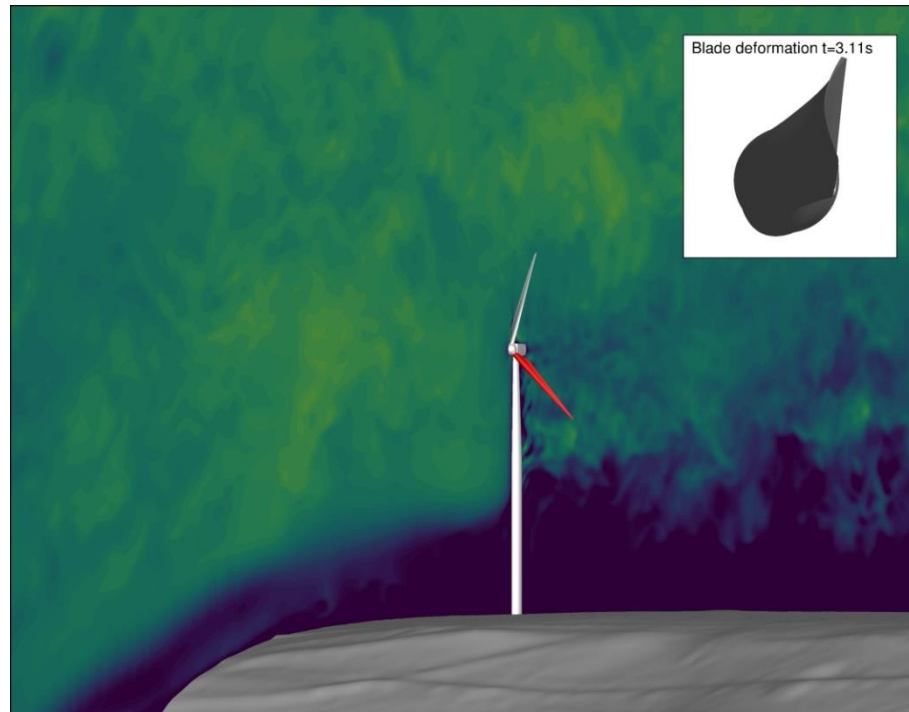
Stable 4D



Stable 6D

P. Letzgus

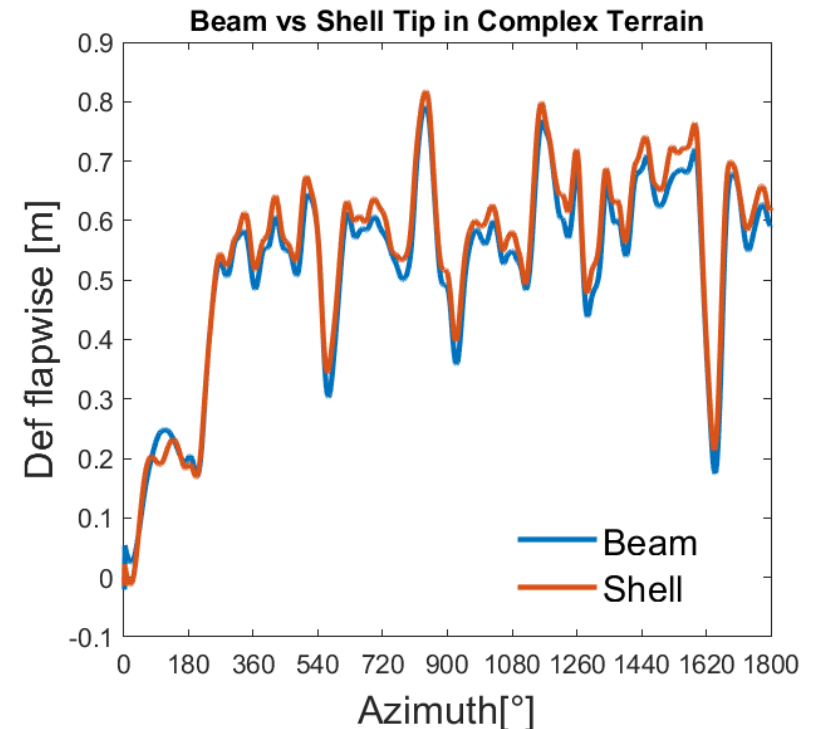
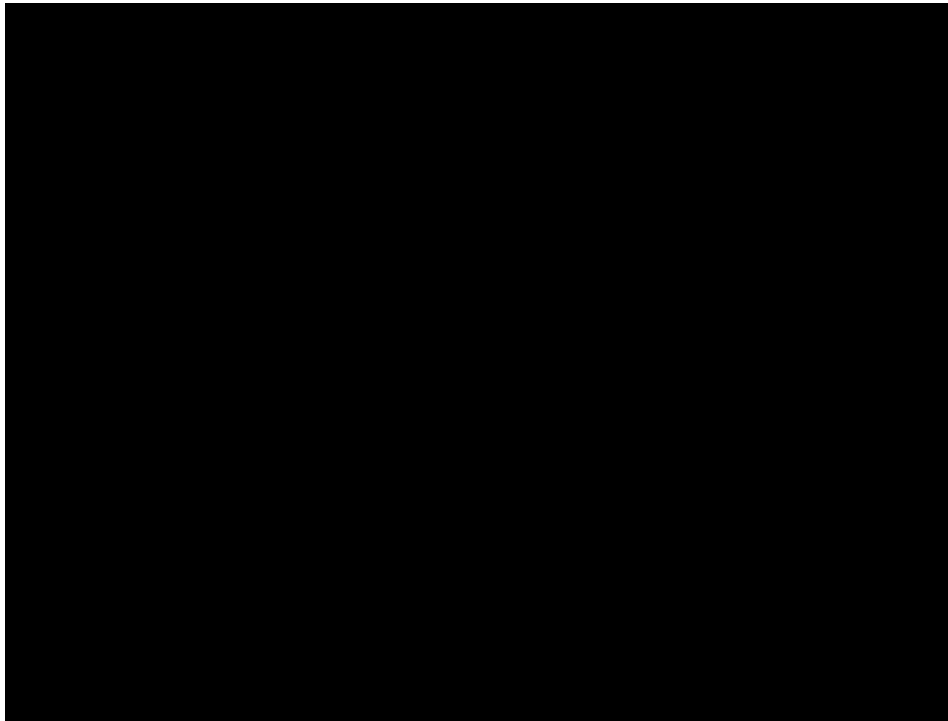
Aeroelasticity Flexible Turbine in Atmospheric Inflow and Complex Terrain



Tip deformation shows similar shape, but shell model predicts up to 5% more deformation. This is directly affecting Thrust and Torque

G. Guma / P. Bucher

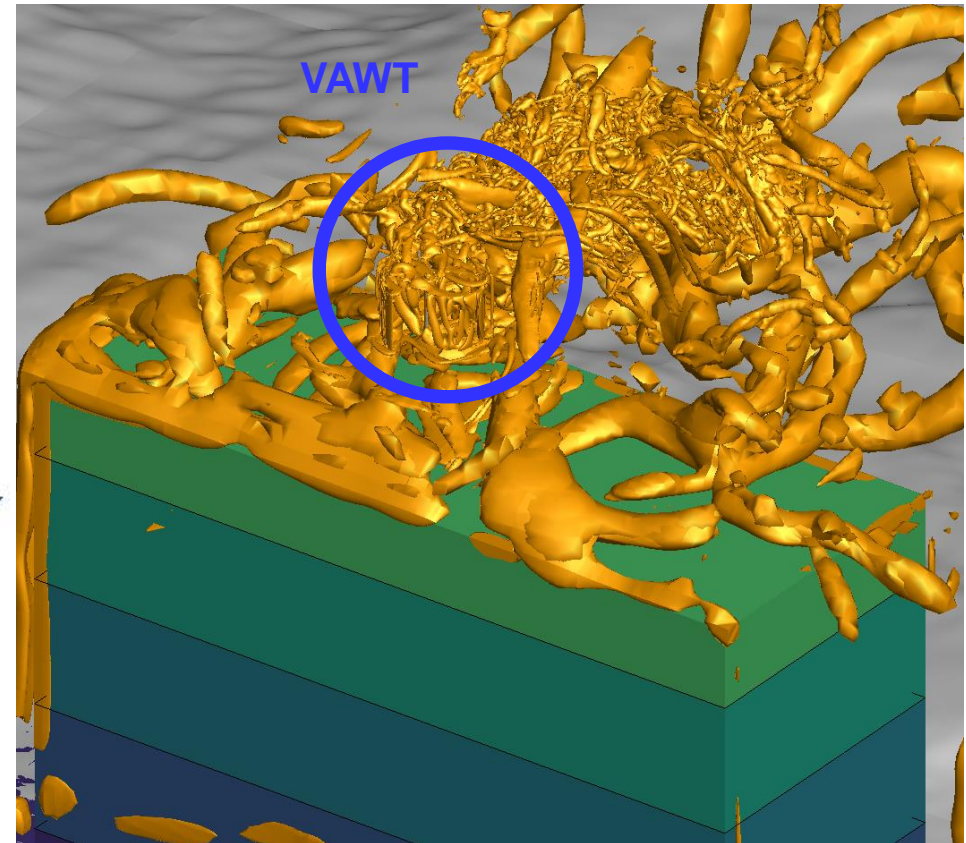
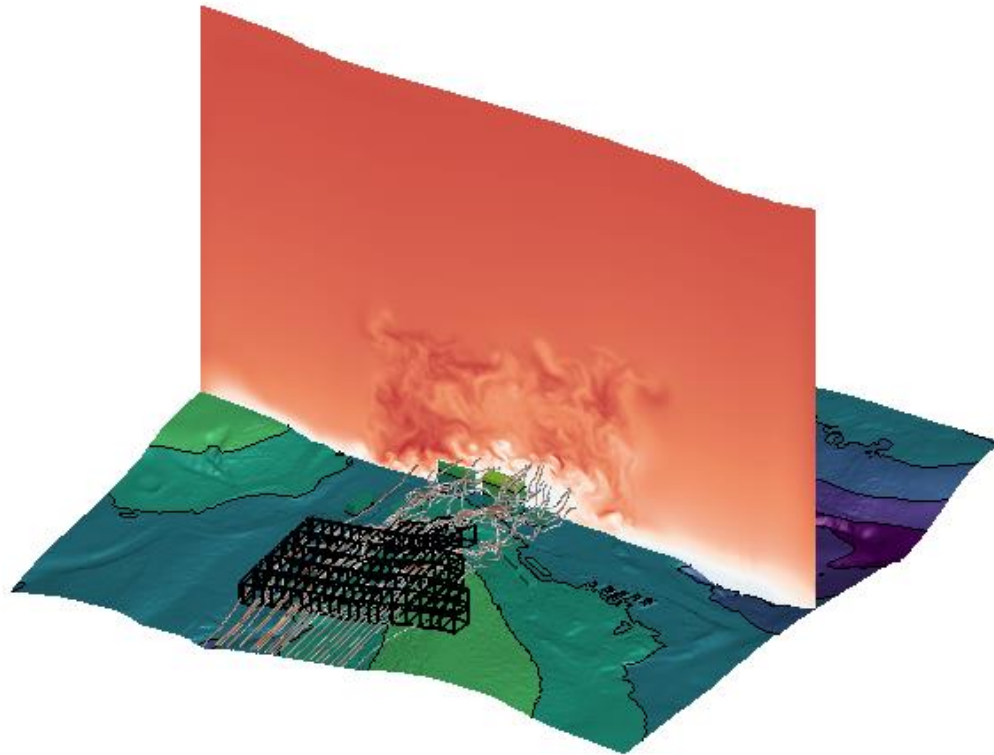
Aeroelasticity Flexible Turbine in Turbulent Inflow and Complex Terrain - Video



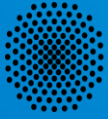
Tip deformation shows similar shape, but shell model predicts up to 5% more deformation. This is directly affecting Thrust and Torque

G. Guma / P. Bucher

Urban Terrain Forrested and Urban Terrain Simulation With VAWT



P. Zamre



University of Stuttgart
Germany
Germany

Thank you!



Thorsten Lutz

e-mail lutz@iag.uni-stuttgart.de

phone +49 (0) 711 685- 63406

University of Stuttgart

Institute of Aerodynamics and Gas Dynamics
Pfaffenwaldring 21, 70569 Stuttgart

Acknowledgement

Results shown in this presentation stem from the following national research projects funded by the German Federal Ministry for Economic Affairs and Energy (BMWi):

WINSENT, TREMAC, KonTest, ActiQuieter, IndianaWind, Schall_KoGe

